

35 TYPING REPERFORATOR

GENERAL DESCRIPTION AND PRINCIPLES OF OPERATION

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Figure 1 - 35 Typing Reperforator

1. GENERAL DESCRIPTION (Fig. 1)

1.01 This section is reissued to include as a standard a complete general description and basic principles of operation of the 35 Typing Reperforator. Changes or additions to the previous issue are indicated by marginal arrows.

1.02 The 35 Typing Reperforator is an electro-mechanical device which records message data on one inch wide paper tape, both as printed characters, or graphics, and as combinations of code perforations. The message data is received from a signal line in the form of an electrical signaling code, which is translated into the necessary mechanical motions to type and perforate information on the tape. The code and feed holes are fully perforated, and the printed characters are typed between the feed hole perforations. The printed characters simplify tape handling by eliminating the need of reading the code perforations.

1.03 The characters perforated in the tape are six positions in advance of the printed characters. This should be considered when severing the tape or inserting it into a transmitter distributor. The end of the tape should include all the printed characters in the message, and the first printed character of the message must be preceded by at least six sets of code perforations.

1.04 The equipment is designed to receive messages transmitted in the eight level American Standard Code for Information Interchange (ASCII). See the applicable section for a detailed explanation of the code.

1.05 Graphic characters (such as A, B, C, 1, 2, 3, #, *, %, &) are printed in black ink. Control characters (such as BELL, EOA, XOFF, SOM, EOT, etc.) are printed in their complementary figures or numeral graphic, but in red ink. This, as well as all other operations of the equipment, is accomplished automatically in response to signal line input while the equipment is in operation.

1.06 Code signals are applied to a magnet associated with a selector mechanism, which interprets the signals and initiates controls of the motion involved in typing and perforating. The magnets are operated on an 0.500 ampere circuit from a selector magnet driver in an associated electrical service unit but they may be wired to operate on either 0.020 or 0.060 ampere current from the DC signal line. A range finder is used to refine the mechanical orientation of the selector mechanism to the signaling code. The selector clutch is held latched by the selector magnets until an incoming signal intelligence initiates typing and perforating by engaging the clutch.

1.07 Motive power is received through a sprocket on the rear of the main shaft.

→ The sprocket is driven by intermediate gearing mechanism of the associated equipment
→ through a jack shaft and gear located just
→ under the main shaft. Rotation of the main shaft is distributed by two all-steel internal expansion clutches, one a selector clutch, the other a function clutch. When engaged, the clutches permit their associated mechanisms to operate at the speed established by rotation of the main shaft. Operating speeds of 60, 75 or 100 words per minute are available through changes in gears.

1.08 The function clutch is tripped by the selector mechanism to initiate transfer of motion from the main shaft to the printing and perforating mechanisms. A transfer mechanism initiates typewheel positioning and printing functions, while a perforator mechanism feeds one inch tape and punches the feed hole and code holes, fully perforated, in a code pattern established by mechanical linkage to the selector mechanism. The tape may be threaded and manually advanced by a hand wheel.

→ 1.09 Perforated code holes correspond to the marking, or current, pulses in the signal circuit. Unperforated portions of the tape correspond to the spacing, or no-current, bits in the intelligence code. Reading from the rear as the tape feeds from the punch block, the code holes represent the 1, 2 and 3 bits of the signal code, the feed hole, and the 4, 5, 6, 7 and 8 bits. Since the eighth bit in the code is always marking, the eighth, or foremost, code hole is always perforated.

→ 1.10 Unless stated to the contrary, references in the text to "left" or "right" indicate the operator's right or left, facing the front of the unit, the selector mechanism at the right and the punch mechanism at the left. In illustrations, unless specifically labeled otherwise, it is assumed that the equipment is being viewed from the front. Pivot points are shown in the drawings by circles or ellipses which are solid black to indicate fixed points and crosshatched to indicate floating points.

→ BASIC UNIT

A. Physical Description (Fig. 2 and 3)

→ 1.11 A cast frame provides mounting facilities for the various mechanisms which comprise the Typing Reperforator. The frame is

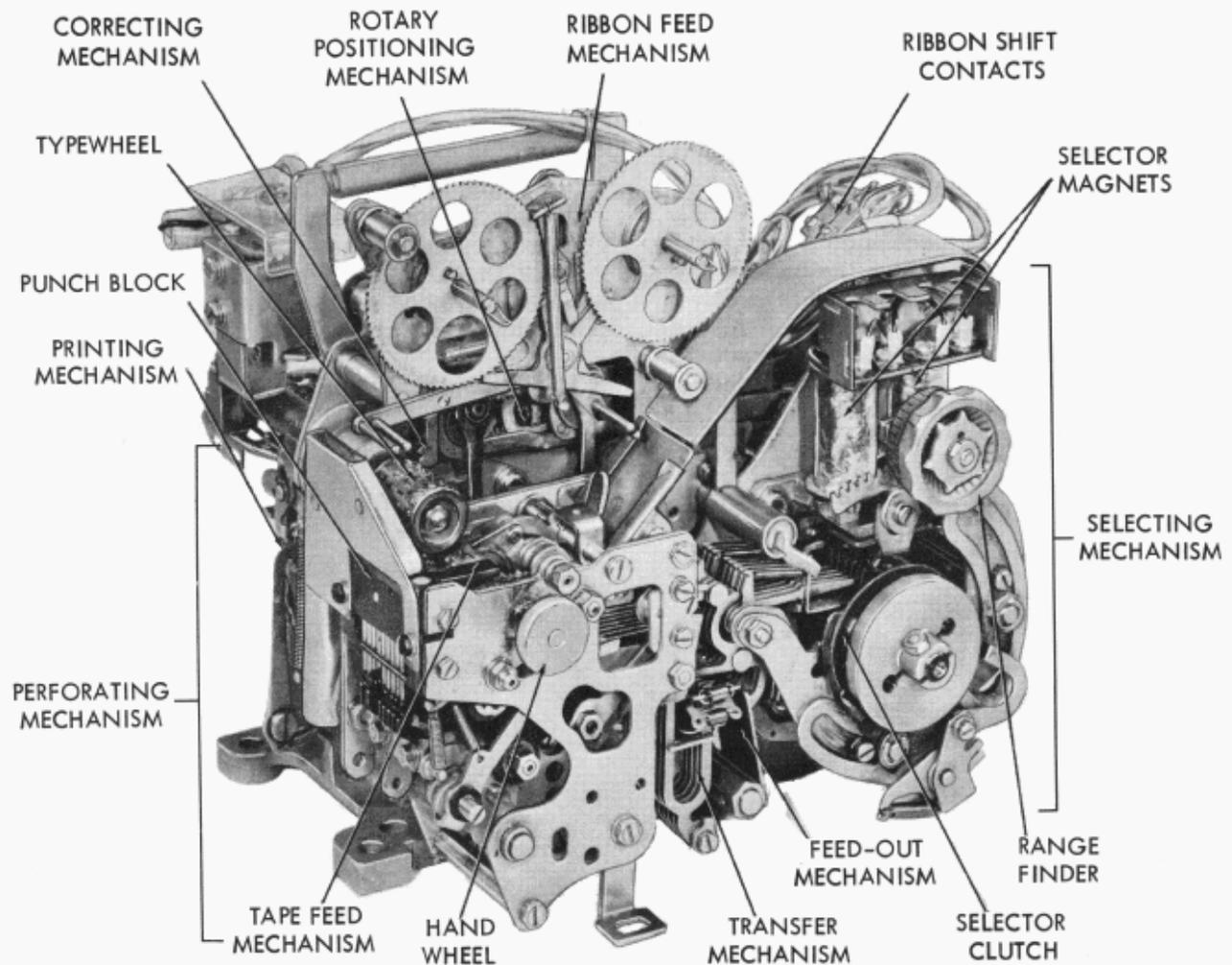


Figure 2 - Typical 35 Typing Reperforator (Left Front View)

in turn mounted on associated equipment through which the necessary electrical and motive power connections are made. For detailed descriptions of associated equipment, motor and cabinet, refer to the appropriate section.

1.12 Mechanical motion from an external source is received through a jack shaft driving a sprocket at the rear of the main shaft. Rotation of the main shaft is transmitted directly to the drum of each of two cam-clutch mechanisms. External changes in speed of the driving power, through a gear shift mechanism or gear changes, will permit changes from 60 to 75 or 100 words per minute in the reperforator operating speed.

1.13 The selecting mechanism is mounted around the front end of the main shaft. It includes a two-coil magnet, a selector cam-clutch and a range finder. By means of the range finder, the selecting mechanism can be adjusted in relation to the signal code.

1.14 The function cam-clutch is tripped by the selecting mechanism. It drives the rocker bail, which transmits power to the printing and perforating mechanisms.

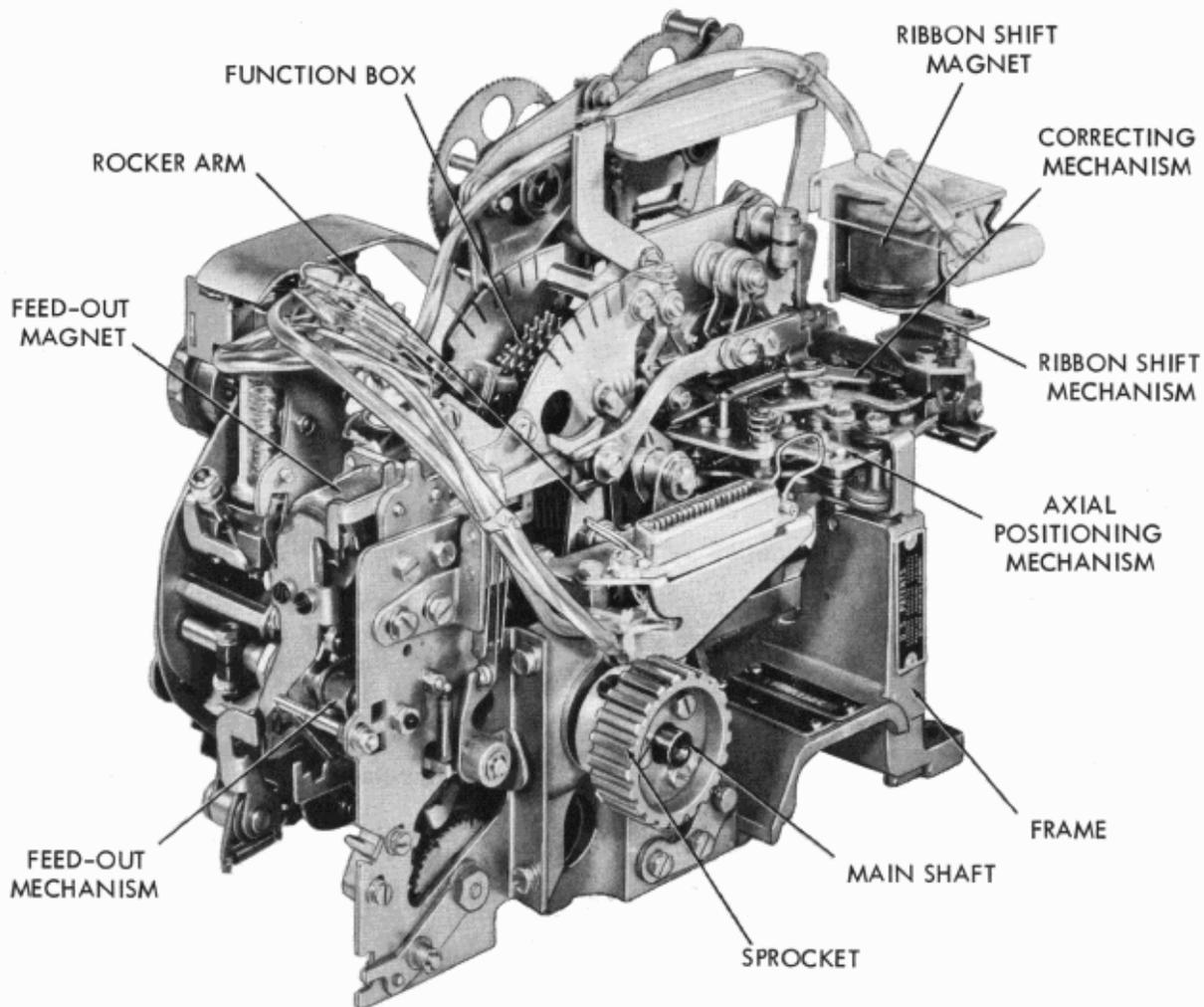
1.15 The rocker bail drives the axial and rotary positioning mechanisms and their respective correcting mechanisms to position the typewheel according to intelligence information

mechanically conveyed from the selecting mechanism through a transfer mechanism. The printing mechanism and ribbon feed mechanism combine to impose the inked image of the selected graphic from the typewheel to its proper position, six characters behind the perforated code to which it corresponds, and between the feed holes on the tape.

→1.16 A ribbon shift mechanism, actuated by ribbon shift contacts on the function box, either permits the ribbon to advance fully to print graphics in black or retards the advance of the ribbon to print the selected character in red. This occurs when the code for a function, rather than a graphic, is transmitted to the reperforator selecting mechanism.

→1.17 The perforator mechanism includes the punch, the tape feed mechanism and a chad disposal chute. The punch accommodates one inch wide tape, perforated ten code characters to the inch with eight level fully perforated code holes. The tape is fed by a feed wheel and die wheel, which indents but does not perforate the tape. The feed hole is perforated in the punch block, simultaneously with the code hole punching and prior to tape feeding. A chad chute mounted over the punch block directs chads produced as the tape is perforated into suitable disposal facilities.

→1.18 The mechanical linkage of selector push levers and punch slide latches and the perforator punch slides interconnects the selec-



→ Figure 3 - Typical 35 Typing Reperforator (Left Rear View)

CODE 1 2 3 4 5 6 7 8	CHARACTER REPRESENTATION	MEANING OF CHARACTER	CHARACTER PRINTED	COLOR
0 0 0 0 0 0 0 0	§ NULL	Blank	□	Red
0 0 0 0 0 0 0 0	§ SOM	Start of Message		Red
0 0 0 0 0 0 0 0	§ EOA	End of Address	"	Red
0 0 0 0 0 0 0 0	§ EOM	End of Message	#	Red
0 0 0 0 0 0 0 0	EOT	End of Transmission	\$	Red
0 0 0 0 0 0 0 0	WRU	Who Are You	%	Red
0 0 0 0 0 0 0 0	RU	Are You	&	Red
0 0 0 0 0 0 0 0	BELL	Bell	'	Red
0 0 0 0 0 0 0 0	§ FE ₀	Form Effector	(Red
0 0 0 0 0 0 0 0	TAB	Horizontal Tabulation)	Red
0 0 0 0 0 0 0 0	LF	Line Feed	*	Red
0 0 0 0 0 0 0 0	VT	Vertical Tabulation	+	Red
0 0 0 0 0 0 0 0	FORM	Form Feed	,	Red
0 0 0 0 0 0 0 0	RETURN	Carriage Return	-	Red
0 0 0 0 0 0 0 0	§ SO	Shift Out	.	Red
0 0 0 0 0 0 0 0	§ SI	Shift In	/	Red
0 0 0 0 0 0 0 0	§ DC ₀	Device Control	0	Red
0 0 0 0 0 0 0 0	§ X On	Transmitter On	1	Red
0 0 0 0 0 0 0 0	R1 On (TAPE)	Receiver On	2	Red
0 0 0 0 0 0 0 0	X Off	Transmitter Off	3	Red
0 0 0 0 0 0 0 0	R1 Off (TAPE)	Receiver Off	4	Red
0 0 0 0 0 0 0 0	§ ERR	Error	5	Red
0 0 0 0 0 0 0 0	§ SYNC	Synchronization Character	6	Red
0 0 0 0 0 0 0 0	§ LEM	Logical End of Medium	7	Red
0 0 0 0 0 0 0 0	§ S ₀	Information Separators	8	Red
0 0 0 0 0 0 0 0	§ S ₁	Information Separators	9	Red
0 0 0 0 0 0 0 0	§ S ₂	Information Separators	:	Red
0 0 0 0 0 0 0 0	§ S ₃	Information Separators	;	Red
0 0 0 0 0 0 0 0	§ S ₄	Information Separators	<	Red
0 0 0 0 0 0 0 0	§ S ₅	Information Separators	=	Red
0 0 0 0 0 0 0 0	§ S ₆	Information Separators	>	Red
0 0 0 0 0 0 0 0	§ S ₇	Information Separators	?	Red
0 0 0 0 0 0 0 0	§ CNFM	Conformation	\	Red
0 0 0 0 0 0 0 0	ALT MODE	Escape (For Communicators)]	Red
0 0 0 0 0 0 0 0	§ ESC	Escape (For Data Processing)	↑	Red
0 0 0 0 0 0 0 0	RUB OUT	Delete (All Bits Marking)	←	Red

NOTE: Characters marked § have no associated keytop on 35 keyboards.

→ Figure 4 — 8-Level Data Interchange Code Language (Controls) for Two Color Tape Printers

→ marking. For even parity code transmission, the eighth bit may be either marking or spacing, so that the number of marking bits in the transmitted code is always an even number. (See Fig. 5.) The intelligence bits are preceded by a start bit (always spacing) and are followed by two stop bits (always marking). Thus each combination consists of 11.0 units of time (referred to as an 11.0 unit transmission pattern). The start and stop bits ensure synchronization between the transmitting and receiving equip-

ment by bringing the receiving equipment to a complete stop at the end of each combination. The marking condition of the eighth bit further enlarges the marking interval at the end of each code combination transmitted.

2.07 The code representations for the graphics U and * are illustrated in Fig. 5. In these combinations, alternate marking and spacing condition for the intelligence bits are required.

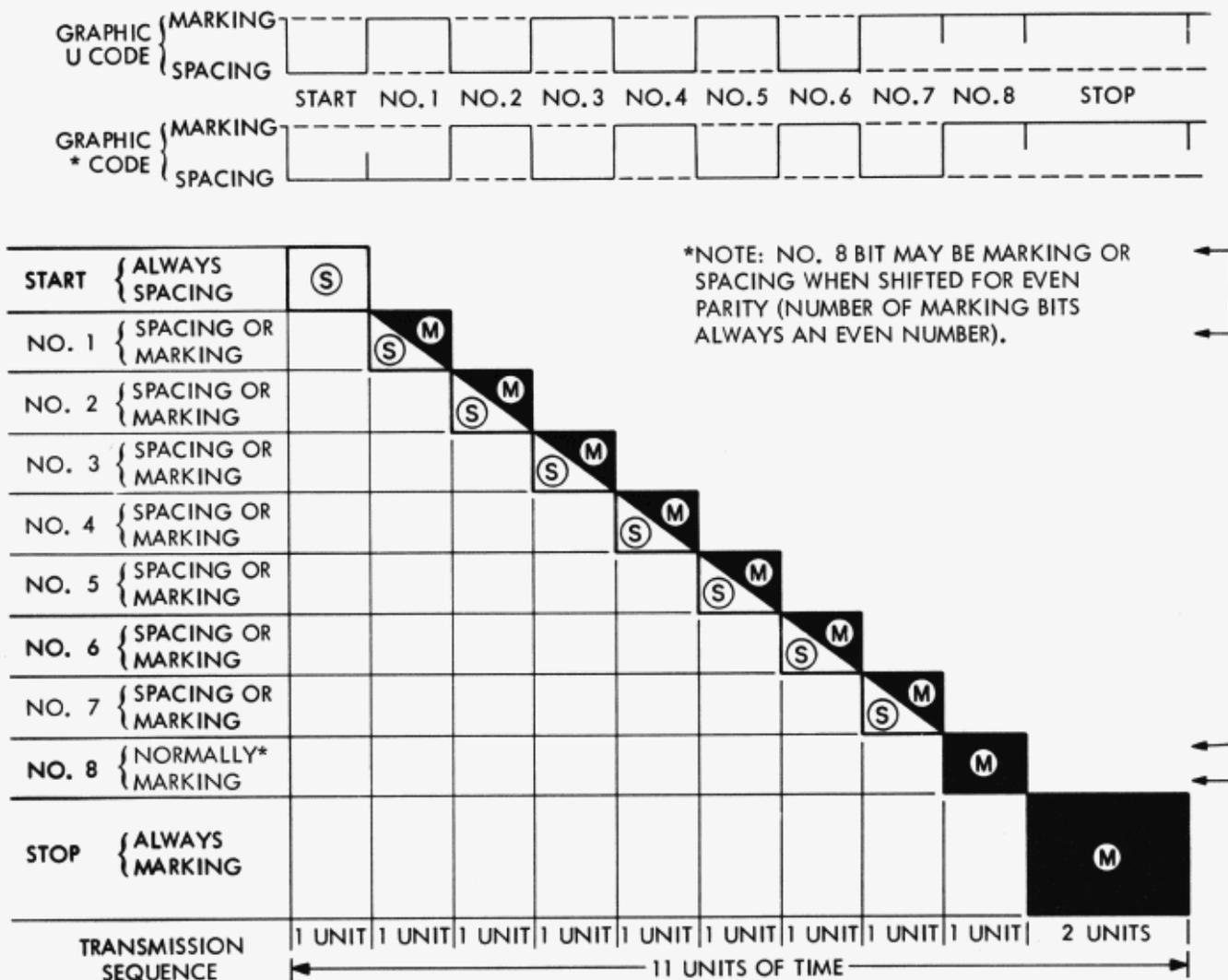


Figure 5 - Signaling Code

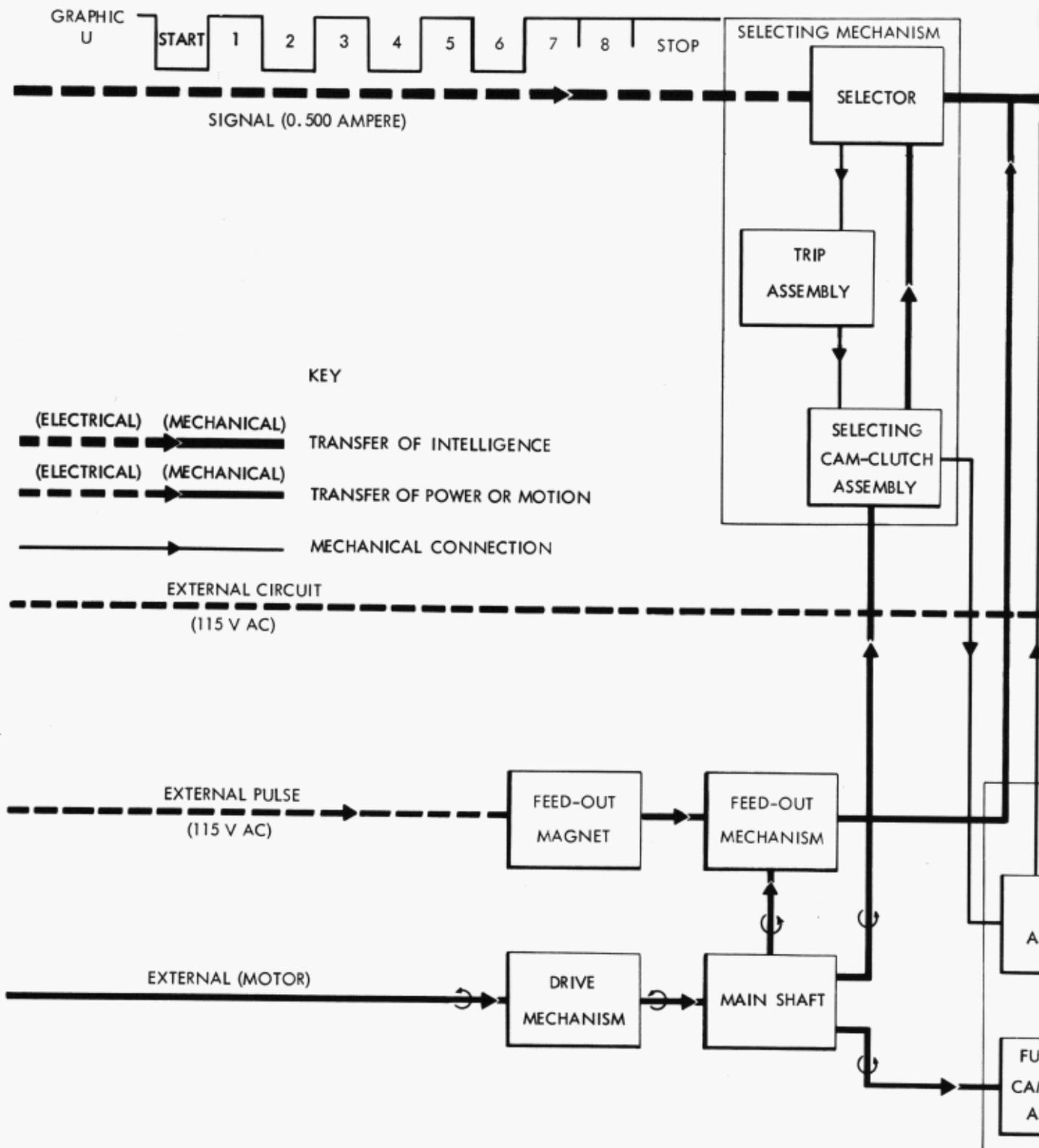
2.08 The total number of permutations of an eight level, 11 unit code (with the eighth level always marking) is two to the seventh power, or 128. Specific character and function representations may vary with equipment. The character (graphic) and functions commonly represented on associated keyboards and typing units are illustrated in Fig. 6. Function representations which are blank are unassigned. For a more complete discussion of the signaling code, refer to the applicable section.

2.09 Marking bits in the intelligence code are represented by holes and spacing bits by the absence of holes. The eighth code level

is always marking (perforated) in present applications but may be either marking or spacing for even parity codes. The row of smaller holes between the third and fourth levels are tape feed holes and do not enter into the code permutation.

GENERAL OUTLINE OF OPERATION (Fig. 7)

2.10 The relationship of the operation mechanisms of the 35 Typing Reperforator are illustrated in the block diagram (Fig. 7). Rotary motion from an external source is applied to the main shaft through a sprocket driven



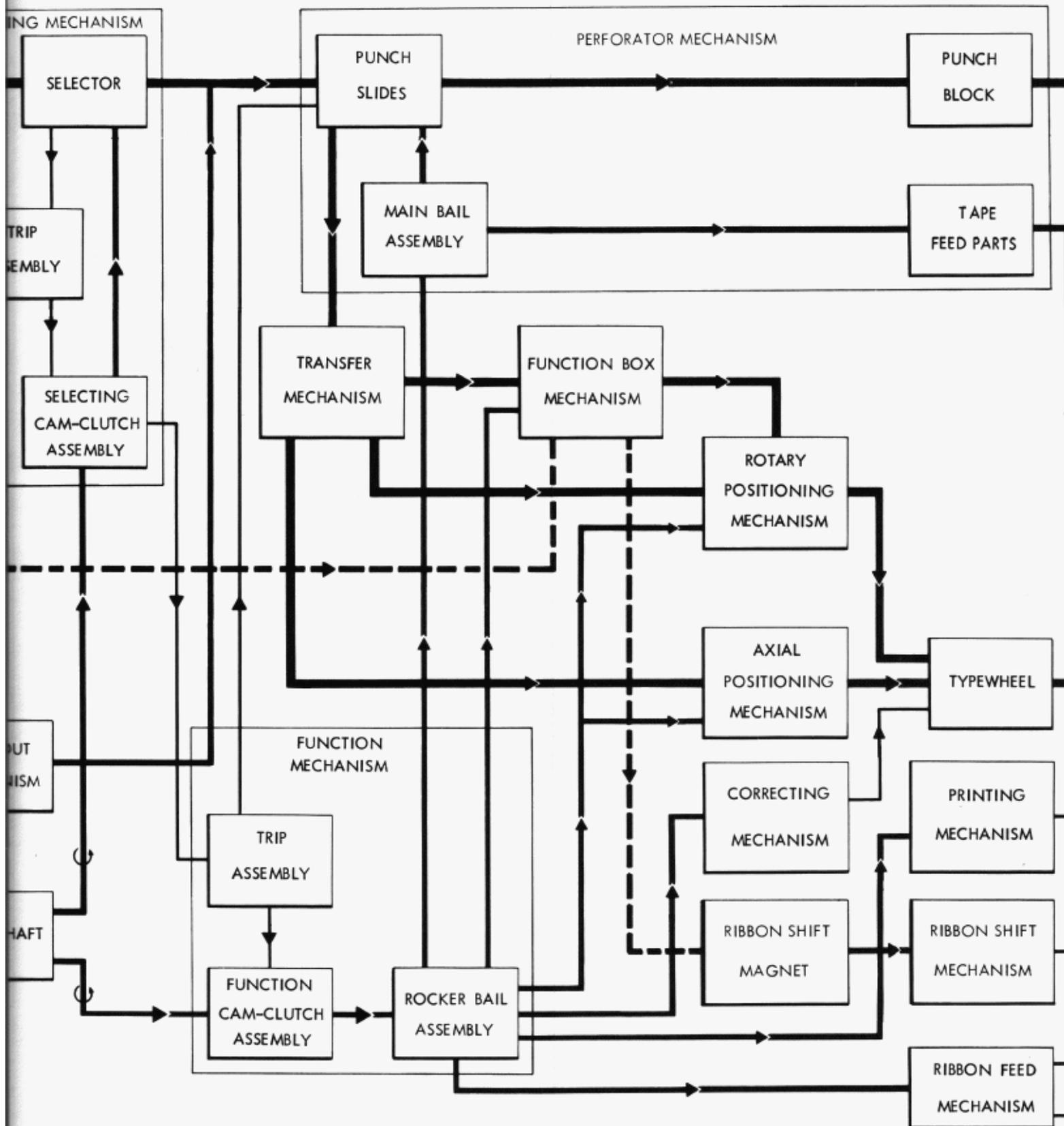


Figure 7 - Block Diagram of

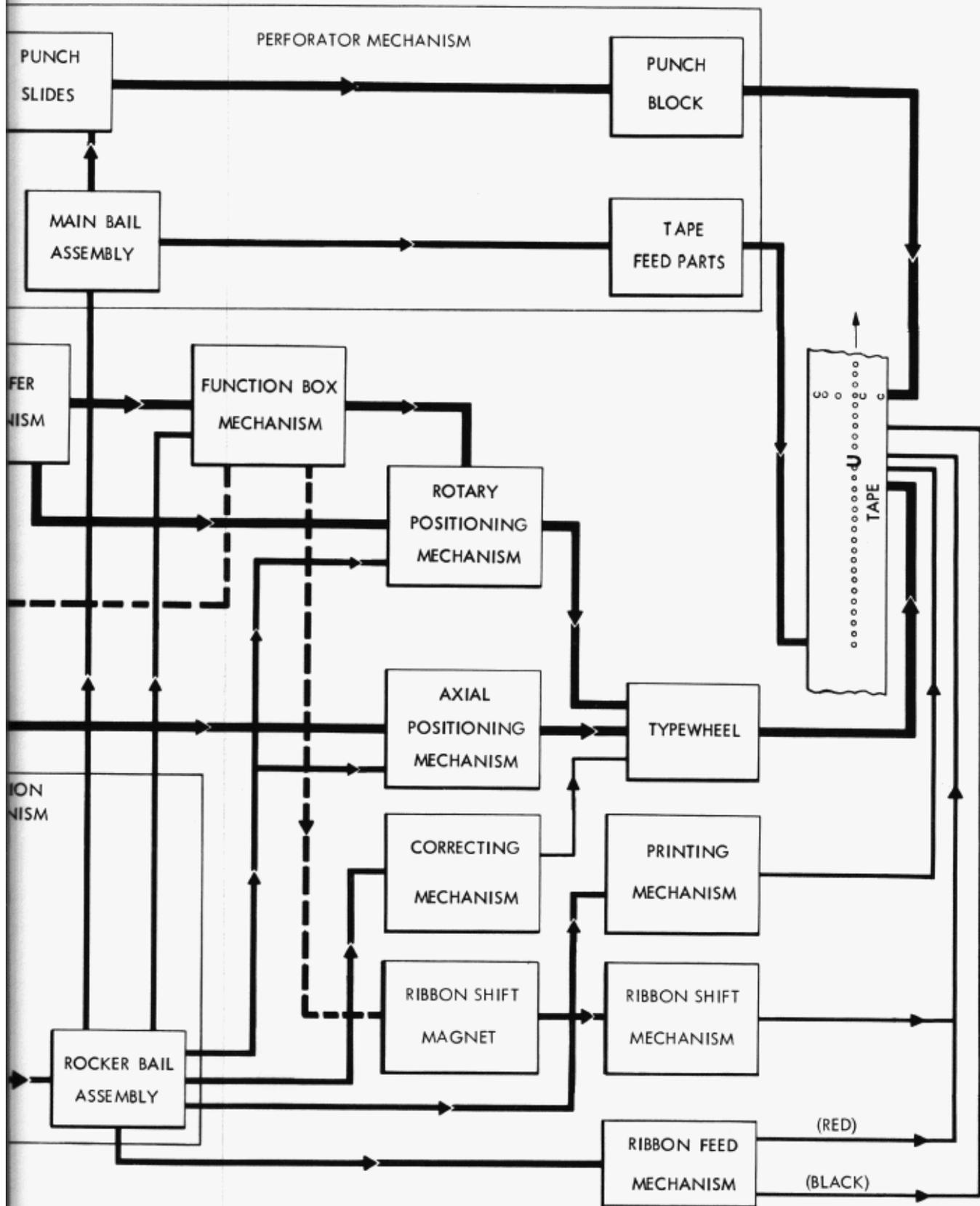


Figure 7 - Block Diagram of 35 Typing Reperforator

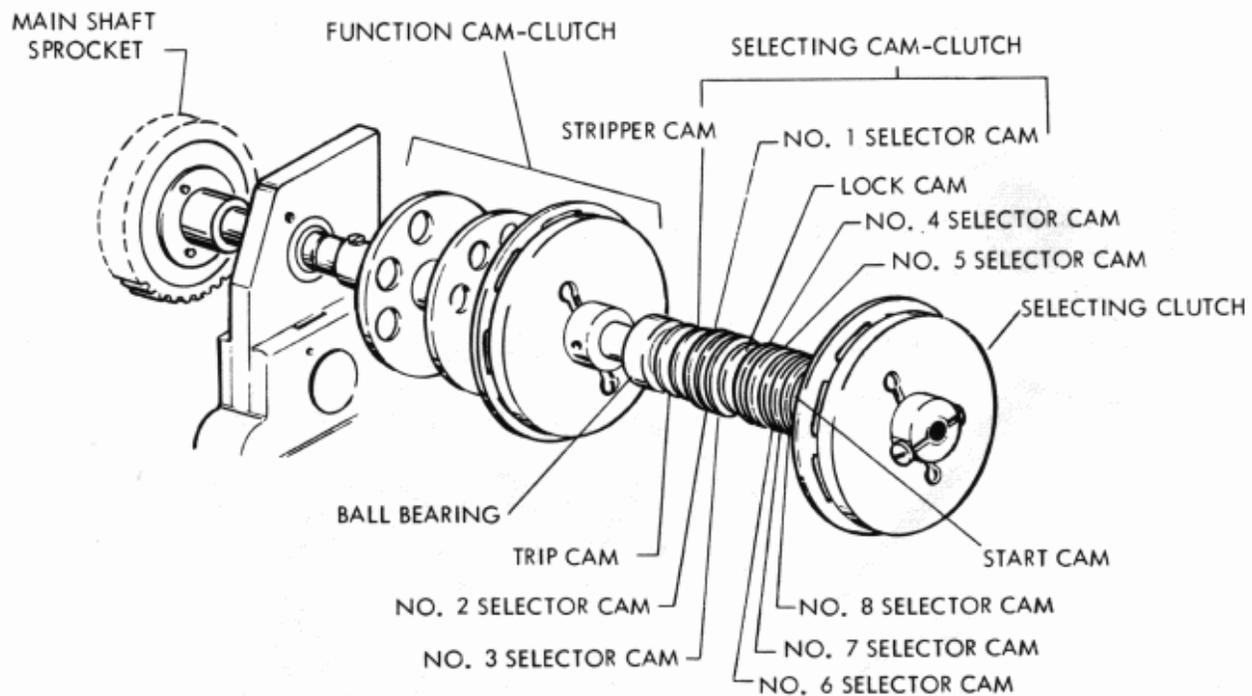


Figure 8 - Main Shaft

the cam-clutch actuates the function trip assembly. The latter trips the function cam-clutch to operate the printing and perforating mechanisms. The selecting cam-clutch is then disengaged and remains inoperative until the next code combination is received.

2.12 The function cam-clutch, driven by the main shaft, imparts motion to the rocker bail throughout the function cycle. The rocker bail transfers the motion to the perforating mechanism, the positioning mechanisms, the tape feed mechanism and the printing mechanism.

2.13 The transfer mechanism, having received their arrangement from the selector, causes positioning of the axial and rotary positioning mechanisms, which select the type-wheel character to be printed.

2.14 The punch slides, having received their arrangement from the selector, cause the punch pins to perforate code holes in the tape corresponding to the code pulses received by the selecting mechanism. Late in the function cycle, the tape feed parts advance the tape one character space. The function cam-clutch is then disengaged and remains stationary until

again tripped by the selecting cam-clutch or by the tape feed-out mechanism. The operations of the reperforator may overlap if the code combinations are being received fast enough. For example, while the perforating mechanism is punching the code combination, advancing the tape and the printing mechanism is printing, the selecting mechanism may be processing the next code combination.

MOTION (Fig. 8)

2.15 Rotary motion from an external source is received by the main shaft through a timing belt, which operates continuously at a constant speed as long as the unit is under power. Selecting and function cam-clutch distribute this motion to the selecting and function mechanisms as described below.

SELECTION

A. General

2.16 The selecting mechanism, made up of a selector (Par. 2.22), a clutch trip assembly (Fig. 9) and a cam-clutch (Fig. 8), translates the signaling code combinations into mechanical arrangements which govern tape

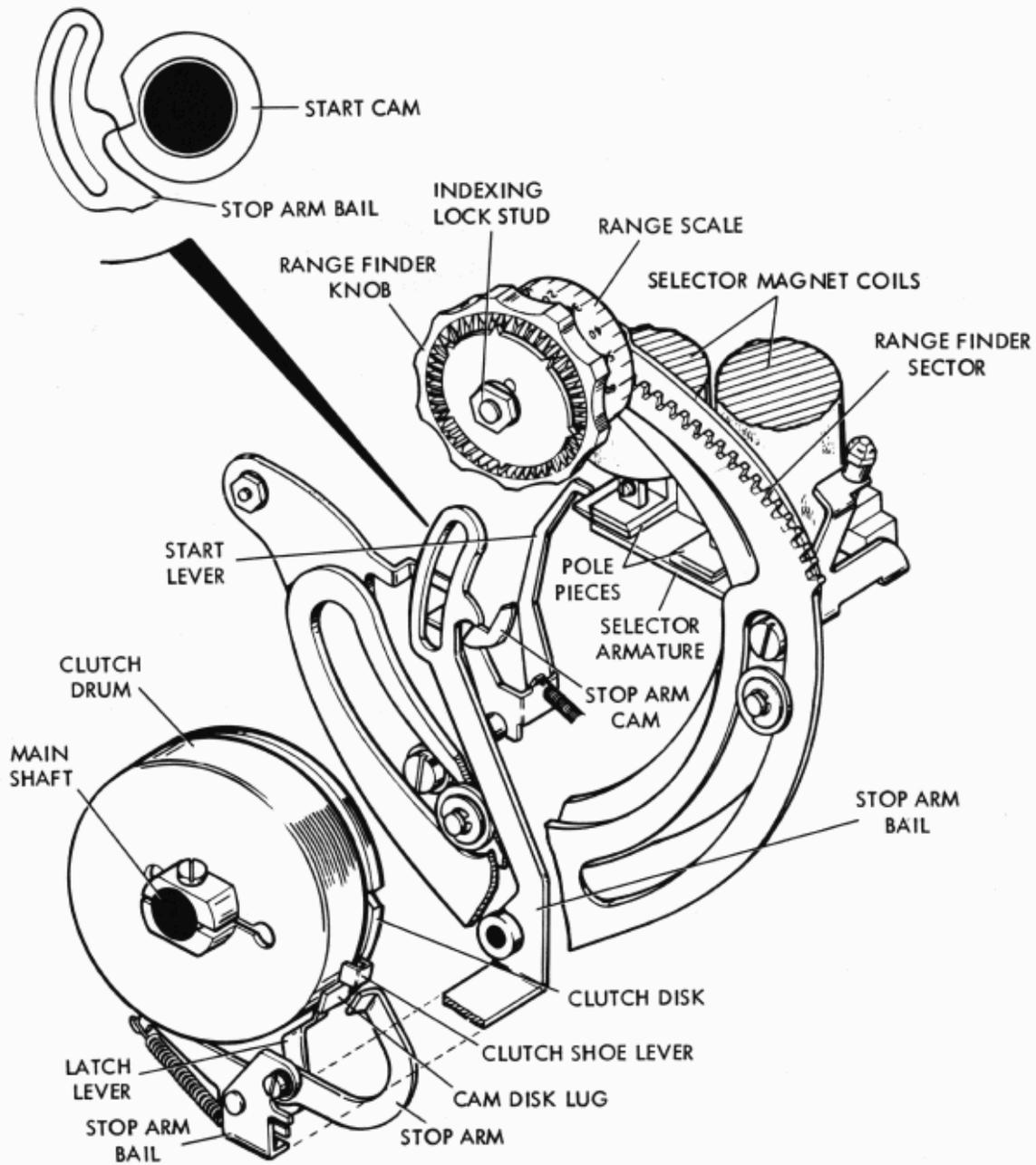


Figure 9 - Range Finder and Selecting Cam-Clutch Assembly

printing and perforation. The electrical pulses comprising each code combination are applied to a magnet of the selector. The magnet, through an armature, controls the clutch trip assembly and the parts associated with translation. The cam-clutch transfers timed motion to the selector and also trips the function cam-clutch. By means of a range finder assembly (Fig. 9), the selecting mechanism can be adjusted to sample the code bits at the most favorable time for optimum operation. The mechanical arrangements produced by the selecting mechanism are passed on through the transfer mechanism to control the positioning and printing mechanisms (Par. 2.28) and through the punch slides to control the perforating mechanism (Par 2.24).

B. Reception and Translation

Selecting Cam-Clutch and Trip Assembly (Fig. 8 and 9)

2.17 The selecting cam-clutch includes (from right to left in Fig. 8) the clutch, the start cam, the eighth, seventh, sixth, fifth and fourth pulse cams, the lock cam, the third, second and first pulse cams, the stripper cam and the trip cam. During the time in which the signal line current is closed (marking), the selector magnet coils are energized and hold the selector armature up against the magnet pole pieces (Fig. 9). In this position, the armature blocks the start lever, and the cam-

clutch is held stationary between the stop arm and latch lever.

2.18 When a code combination is received, the start bit (spacing) de-energizes the magnet, and the selector armature under tension of its spring moves down out of the way of the start lever. The start lever turns clockwise under spring pressure and moves the stop arm bail into the indent of the start cam (Fig. 9). As the stop arm bail rotates about its pivot point, the attached stop arm is moved out of engagement with the clutch shoe lever. The selecting cam-clutch engages and begins to rotate counterclockwise. The stop arm bail immediately rides to the high part of the cam, where it remains to hold the start lever away from the armature while the intelligence bits of the code are received and processed by the selector (Par. 2.22 to 2.24).

2.19 When the stop bit at the end of the code combination is received, the armature is pulled up and blocks the start lever. Thus the stop arm bail is prevented from dropping into the low part of its cam, and the attached stop arm is held in position to stop the clutch stop lever. When the clutch shoe lever strikes the stop arm, the inertia of a cam disk causes it to continue to turn until its lug makes contact with the clutch shoe lever. At this point, a latch lever drops into an indent in the cam disk, and the clutch is held disengaged until the next code combination is received.

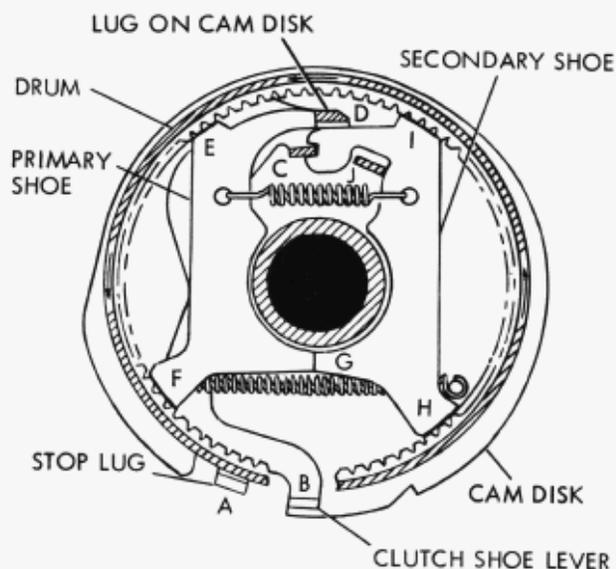


Figure 10 - Clutch, Engaged

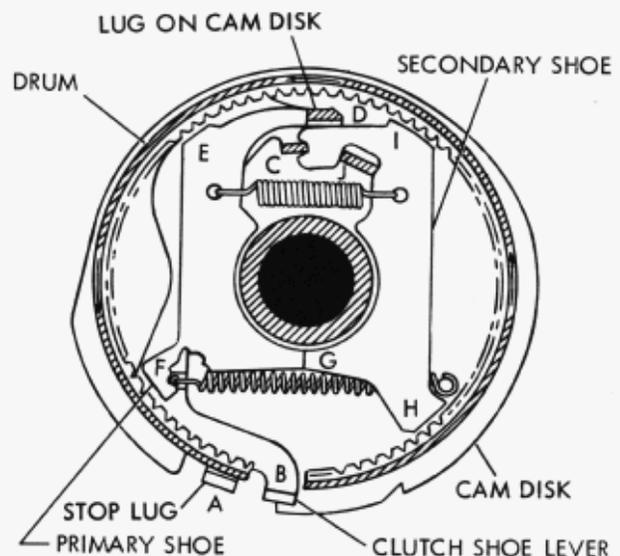


Figure 11 - Clutch, Disengaged

Clutch Operation (Fig. 10 and 11)

2.20 The clutch drum is attached to and rotates in unison with the main shaft (Fig. 8). In the disengaged position, as shown in Fig. 11, the clutch shoes do not contact the drum, and the shoes and cam disk are held stationary. Engagement is accomplished by moving the stop arm (Fig. 9) away from the clutch and thus releasing stop lug A and the lower end of shoe lever B (Fig. 10). The upper end of lever B pivots about its ear C, which bears against the upper end of the secondary shoe, and moves its ear D and the upper end of the primary shoe toward the left until the shoe makes contact with the notched inner surface of the rotating drum at point E. As the drum turns counterclockwise, it drives the primary shoe downward so that it again makes contact with the drum at point F. There, the combined forces acting on the primary shoe cause it to push against the secondary shoe at point G. The lever end of the secondary shoe then bears against the drum at point H. The drum drives this shoe upward so that it again makes contact with the drum at point I. The forces involved are multiplied at each of the preceding steps. The aggregate force is applied through the shoes to the lug J on the clutch cam disk, and the disk and attached cam turn in unison with the drum.

2.21 Disengagement is effected when the lower end of shoe lever B strikes the stop arm (Fig. 9). Lug A and the lower end of the shoe lever are brought together (Fig. 10), and the upper end of lever B pivots about its ear C and allows its other ear D to move toward the right. The upper spring then pulls the two shoes together and away from the drum. The latch lever seats in the indent in the cam disk (Par. 2.19) and the cam is held in its stop position until the clutch is again engaged.

Selector Operation (Fig. 8, 9 and 12)

2.22 The selector assembly consists primarily of two magnet coils (Fig. 9), an armature and associated bails, levers and latches (Fig. 12). Eight linkages, each of which consists of a selecting lever, a push lever and a punch slide latch, link the selector cam with the punch slides. Since the linkages are identical, only the No. 4 is shown in its entirety in Fig. 12. As the selecting bits of the code combination are applied to the magnet, the cam actuates the selecting levers. When a spacing bit is received, a marking lock lever is blocked by the end of the armature, and a spacing lock lever swings

to the right above the armature and locks it in the spacing position until the next signal transition occurs. Extensions on the marking lock lever prevent the selecting levers from following their cams. When a marking bit is received, the spacing lock lever is blocked by the end of the armature, and the marking lock lever swings to the right below the armature and locks it in the marking position until the next signal transition occurs. During this marking condition, the selecting levers are not blocked by the marking lock lever extensions, but are permitted to move against their respective cams. The selecting lever that is opposite the indent in its cam, while the armature maintains a marking condition, swings to the right, or selected, position, and the end of an associated push lever falls off a step on the selecting lever.

2.23 As the cam rotates, the selecting levers, together with any selected push levers, are moved to the left by the high part of their respective cams, where they remain until the next code combination is received. The unselected push levers remain to the right. When the next code combination is received, a selector reset bail, lifted by its cam (Fig. 12), strips the selected push levers from the selecting levers, and the push levers are returned to the right by their springs.

2.24 The selected push levers, in moving to the left, rotate associated punch slide latches counterclockwise (Fig. 12). Just before the eighth push lever is selected, the selecting cam through the function trip assembly causes the perforator reset bail to release the punch slides (Par. 2.28). The unselected latches retain their associated slides to the right, while the selected latches permit their slides to move to the left under spring tension. During the latter part of the function cycle, the reset bail returns the punch slides to their unselected position (Par. 2.63). The latches under spring tension return to their unselected position when the push levers are repositioned at the beginning of the next selecting cycle.

C. Orientation (Fig. 9)

2.25 For optimum performance, the selecting mechanism should be adjusted to sample the signaling code bits at the most favorable time. To make this adjustment, the operating margins are established through the range finder, which provides a means of varying the time of sampling. The obtaining of this optimum setting is referred to as orientation.

2.26 When the range finder knob (Fig. 9) is pushed inward and rotated, its attached range finder gear moves the range finder sector (which supports the stop arm bail, stop arm and latch lever) either clockwise or counter-clockwise about the selector cam-clutch. This changes the angular position at which the selector cam-clutch stops with respect to the marking and spacing lock levers. When an optimum setting is obtained, the range finder knob is released. Its inner teeth engage the teeth of the indexing lock stud and hold the range finder mechanism in position. The setting may be read on the range scale opposite a fixed index mark.

D. Transfer (Fig. 13)

2.27 The function of the transfer mechanism is threefold:

- a. It provides a path for the signal intelligence from the selector to the associated push bar in the typewheel positioning mechanism,
- b. It provides a path for the signal intelligence from other signal sources to the typewheel positioning mechanism, and

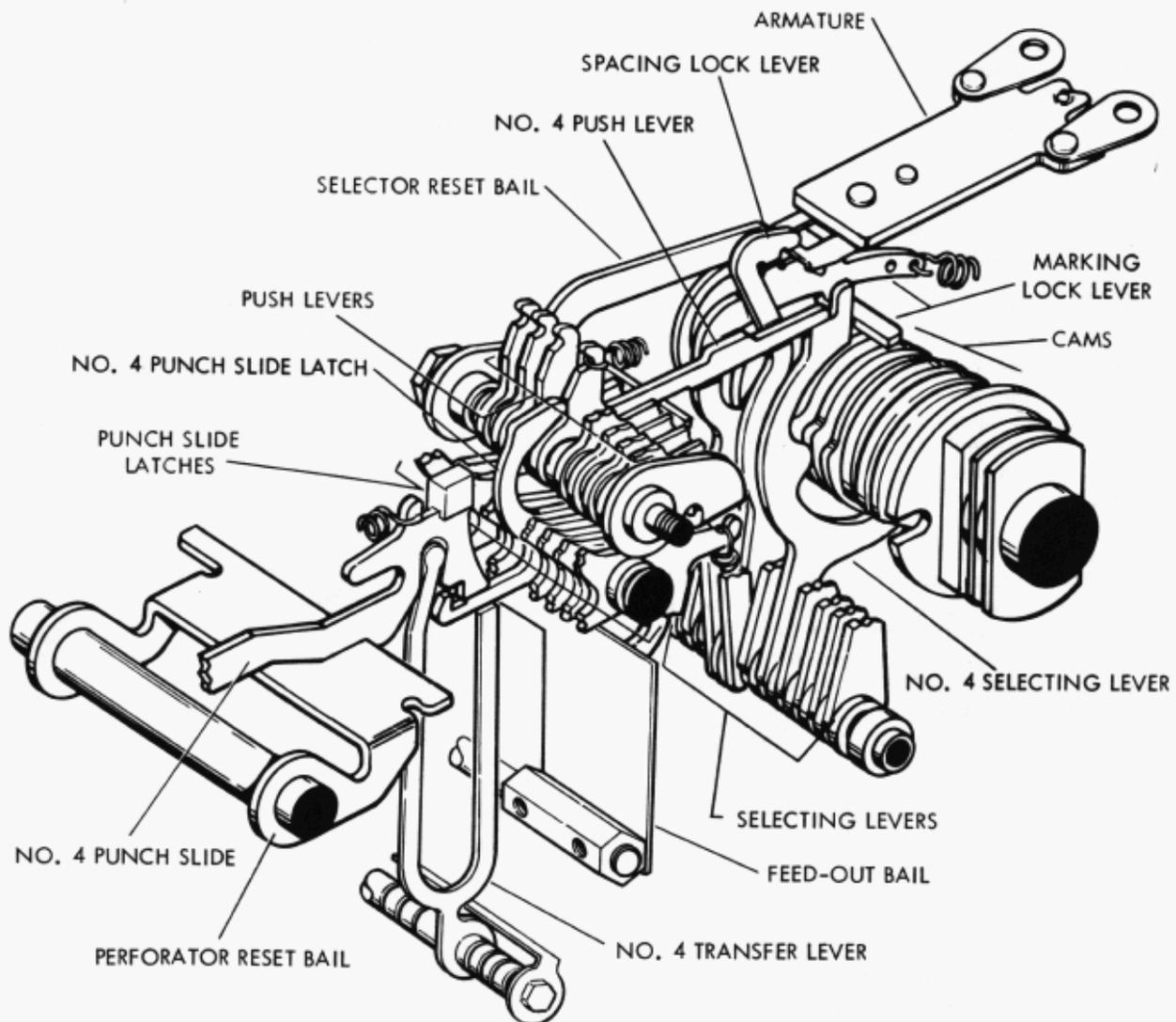


Figure 12 - Selector

- c. It provides a means for setting up the ribbon color shift contacts to condition ribbon for red or black printing.

2.28 The transfer levers engage the punch slides at one end, as illustrated by the No. 4 transfer lever in Fig. 13. The transfer levers all pivot about a common point and, at various distances from this pivot, engage their corresponding transfer beams. The opposite end of the transfer beam is coupled to one arm of a bell crank lever. The opposite arm of the number 1, 2, 3, 4, 5 and 7 bell crank lever engages its associated pushbar. Since the No. 6 and 8 pulses do not control the position of the type-wheel, they do not have an associated pushbar. When a selected punch slide falls forward, the corresponding push bar is raised upwards and into engagement with the rocker bail. An additional extension on the lower end of the latch lever is arranged to engage a bail on the tape feed-out mechanism.

2.29 The No. 6 and 7 bell cranks have an additional arm which controls a transfer contact assembly in the function box. This pair of contacts is used to control the ribbon shift magnet which, in turn controls the color of the printed character. Current is allowed to pass through the contacts when the No. 6 and 7 pulses are opposite polarity, such as No. 6 marking, No. 7 spacing, or No. 6 spacing, No. 7 marking. Current is not allowed to pass when the No. 6 and No. 7 pulses are of the same polarity.

2.30 The bell cranks are provided with an arrangement of projections and slots which either block or permit the entrance of a sensing blade. The function box provides slots for up to six sensing blades which can be coded to respond to any of 256 code combinations. Contact assemblies associated with the sensing blades provide a means of supplying a pulse of between 10 and 14 milliseconds for control purposes with external circuitry.

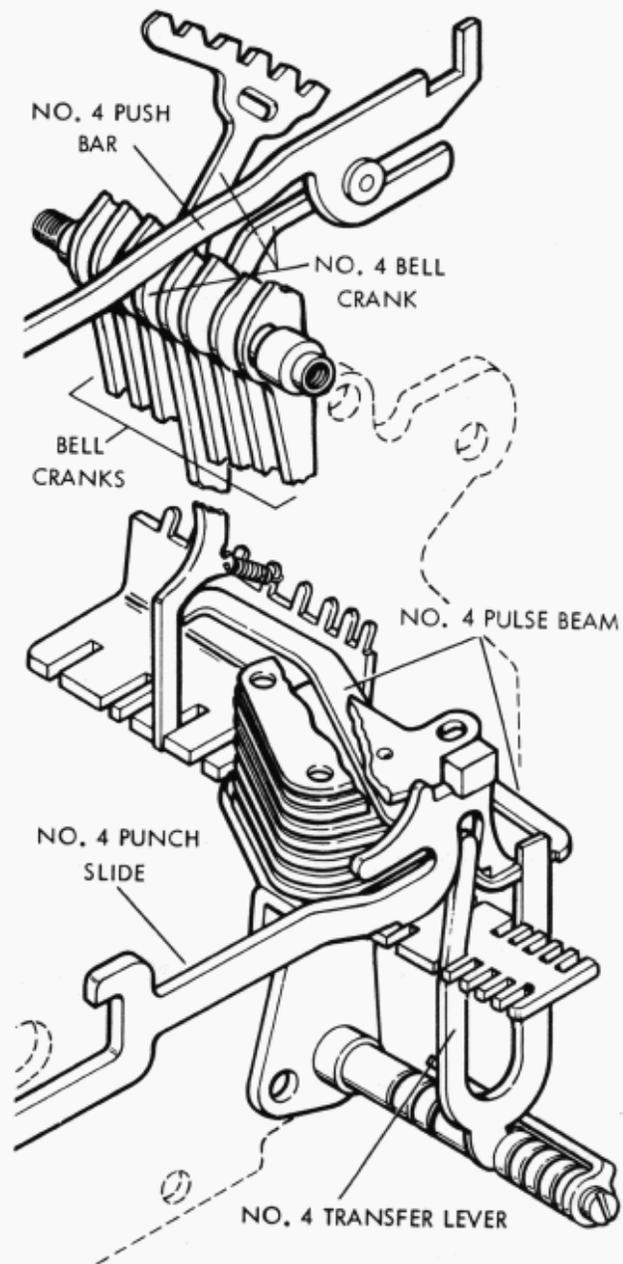


Figure 13 - Transfer Mechanism

MOTION FOR TYPING AND PERFORATING

A. General

2.31 The motion of the main shaft is conveyed to the mechanisms concerned with typing and perforation by the function mechanism, which is comprised of a cam-clutch (Fig. 8), a clutch trip assembly (Fig. 14) and a rocker bail (Fig. 15).

B. Function Cam-Clutch and Clutch Trip Assembly (Fig. 14)

2.32 The trip assembly is shown in its unoperated condition in Fig. 14. A follower lever rides on a function trip cam which is part of the selecting cam-clutch (Fig. 8). Near the

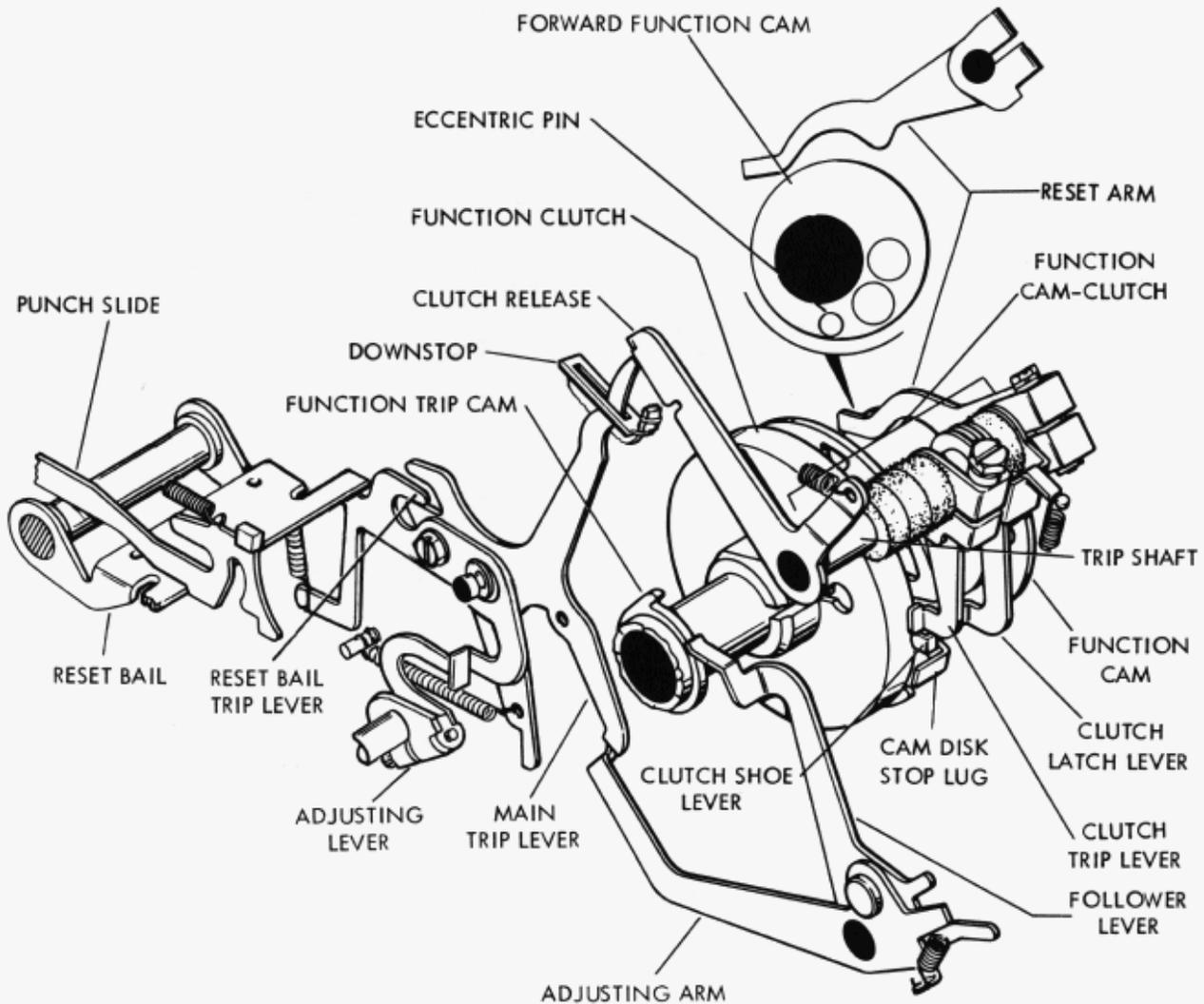


Figure 14 - Function Cam-Clutch and Clutch Trip Assembly

end of the selecting cycle, as the main shaft rotates counterclockwise, the high part of the cam pivots the follower lever (Fig. 14) which, through an attached adjusting arm, rotates a main trip lever counterclockwise. A reset bail trip lever attached to the main trip lever lowers the perforator reset bail and releases the punch slides (Par. 2.63); and an upper arm of the main trip lever moves out of the way of a clutch release, which falls against a downstop and rotates a trip shaft counterclockwise. Immediately, the low part of the trip cam allows the follower lever to return to its unoperated position, and the upper arm of the

main trip lever moves down against the release. When the trip shaft is rotated by the release, it moves an attached clutch trip lever out of engagement with the clutch shoe lever. The clutch engages, and the cam-clutch begins its cycle. The internal operation of the clutch is the same as that of the selector clutch, described in Par. 2.20 and 2.21 of this section.

2.33 About midway through the function cycle, an eccentric pin on the function cam lifts a reset arm, which rotates the trip shaft clockwise. The release is moved up and allows the main trip lever to fall against the adjusting

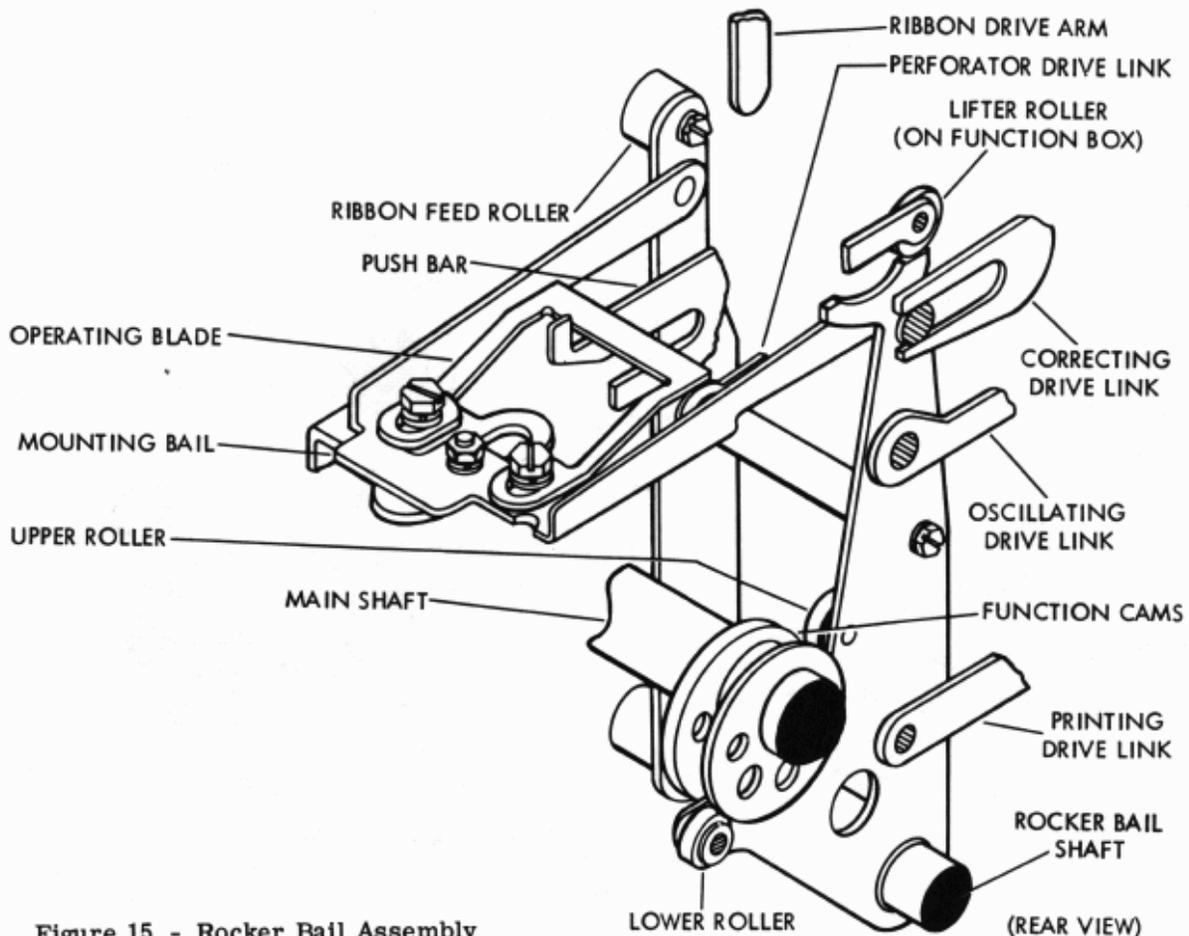


Figure 15 - Rocker Bail Assembly

arm and raise the reset bail. The eccentric pin then moves out from under the reset arm, and the release is permitted to return to its unoperated position against the main trip lever. When the cam-clutch assembly completes its cycle, the clutch shoe lever strikes the trip lever, and the clutch is disengaged.

C. Rocker Bail (Fig. 15)

2.34 The function cam and the rocker bail translate the rotation of the main shaft into simple harmonic motion, which the bail distributes to the following:

- Ribbon feed mechanism
- Perforator
- Correcting mechanism
- Function box
- Printing mechanism

Oscillating assembly

Push bars of the axial and rotary positioning mechanisms.

The bail is shown in its home position in Fig. 15. Each function cycle, the function cams bear against the rollers and cause the bail to rock to the right (as viewed from the rear in Fig. 15) during the first part of the cycle and then back to the home position during the latter part of the cycle.

TYPING

A. General

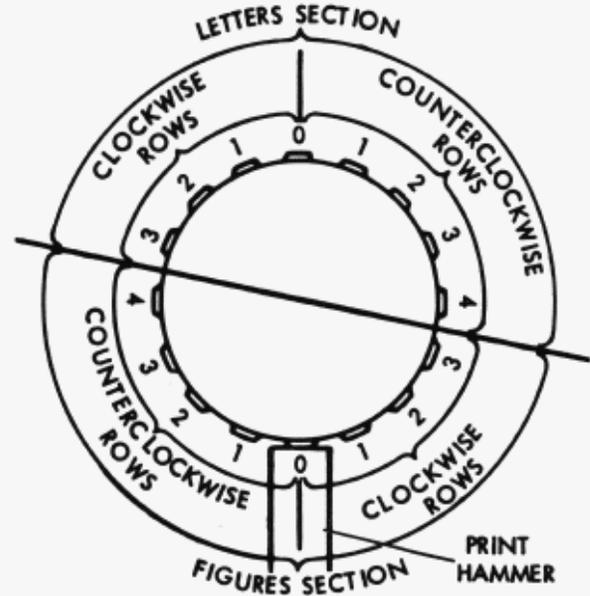
2.35 The characters used to type the received intelligence — letters, figures, and symbols representing various functions — are embossed on the cylindrical surface of the metal typewheel (Fig. 16). During the function cycle, the axial and rotary positioning mechanisms

(Fig. 17 and 19), having received the intelligence from the transfer mechanism, position the wheel so that the character represented by the received code combination is selected. Following typewheel positioning the correcting mechanism (Fig. 17 and 19) accurately aligns the selected character. Then the printing mechanism (Fig. 21), by means of a hammer, drives the tape and inked ribbon against the wheel and imprints the character. A ribbon feed mechanism (Fig. 22) advances the ribbon and reverses its direction of feed when one of two ribbon spools is depleted. Near the end of the function cycle the axial positioning mechanism retracts the typewheel and a ribbon guide so that the last printed character is visible. The letters or the figures code combination sets up an arrangement in the transfer mechanism which permits the function box (Fig. 20) to operate and cause the rotary positioning mechanism to shift the typewheel.

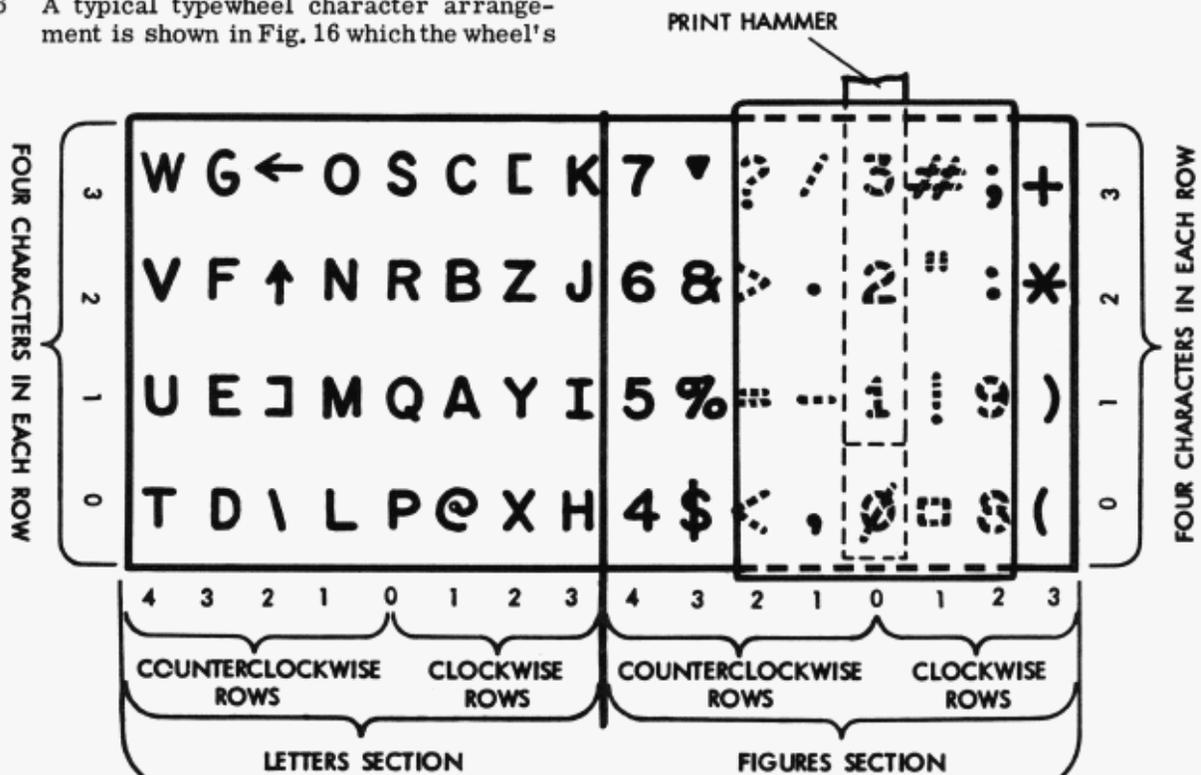
B. Typewheel Positioning

General

2.36 A typical typewheel character arrangement is shown in Fig. 16 which the wheel's



a. FRONT VIEW SHOWING 16 LONGITUDINAL ROWS



b. TOP VIEW SHOWING CYLINDRICAL SURFACE IN A PLANE

Figure 16 - Typical Typewheel Character Arrangement

cylindrical surface is shown rolled out into a plane. There are 16 longitudinal rows, each of which is made up of four characters numbered 0 to 4 from front to rear. The surface is divided into two sections, a letters and a figures, each containing eight rows. The fifth row counterclockwise from the division line in both sections is numbered 0, and there are four rows in one direction from 0 numbered 1 to 4 and designated as counterclockwise rows, and three rows in the other direction numbered 1 to 3 and designated as clockwise rows. It should be noted that the clockwise and counterclockwise modifiers refer to the direction of rotation of the wheel to select the rows and not to their position on the wheel.

2.37 Each printing operation (excluding those devoted to the letters-figures shift) begins and ends with the typewheel in the home position of the section containing the character to be printed, i.e., with the No. 0 character of the No. 0 row at the point of contact of the print hammer. (Actually, inasmuch as the wheel is retracted to show the last printed character (Par. 2.45), the No. 0 character is slightly to the rear, but for this discussion it will be assumed that it is at the point of contact.) During the printing operation the axial and rotary positioning mechanisms, transferring separate but simultaneous motions to the wheel, position it so that the character represented by the received code combination is at the point of contact of the hammer at the time of printing. The rotary mechanism, which is controlled by the No. 3, 4 and 5 selecting elements of the code, revolves the wheel so as to select the proper row; and the axial mechanism, which is governed by the No. 1 and 2 elements, moves it forward and rearward along its axis so as to select the proper character in the row. Rotation of the typewheel to print in either the letters or the figures section is controlled by the No. 7 bit of the code.

2.38 To illustrate the above, if the wheel is in the figures condition, as shown in Fig. 16, and the numeral "0" is to be printed, there is no movement of the wheel during the printing operation, because "0" is already at the point of contact of the hammer. However, if the letter "F" is to be printed, the wheel is first shifted eight rows to the letters home position. Then during the next operation it is rotated three rows counterclockwise and moved forward two characters so that "F" is at the point of contact of the hammer. Printing takes place, and the wheel is then returned to the letters home position.

Rotary Positioning (Fig. 17 and 18)

2.39 The rotary positioning mechanism revolves the typewheel so that the row containing the character to be printed is aligned with the print hammer at the time of printing. Mounted on the front plate, the mechanism includes two eccentric assemblies as shown in Fig. 17 and 18. Each assembly includes a primary shaft, a section of which is formed into a pinion. A secondary shaft, mounted in the primary and offset from its center, forms an eccentric, referred to as the rear eccentric. A portion of the secondary shaft is also a pinion, and a crank pin mounted on its disk-like forward surface forms a secondary, or front, eccentric. Each of the four pinions of the two eccentric assemblies is engaged by the rack of a pushbar: the No. 3 bar engages the right front pinion, the No. 4 engages the left rear pinion and the No. 5 engages the right rear pinion. The left front pinion is engaged by both the letters and the figures pushbar.

2.40 The eccentric assemblies are linked to a typewheel shaft by a drive assembly as shown in Fig. 17. The typewheel is secured to the front of the shaft which is supported by a bearing housing mounted at the left rear of the front plate (Fig. 19). A spur gear which meshes with a typewheel rack rides on the shaft in a bearing housing. The shaft is free to move axially in the housings and the spur gear, but flats in its circumference which bear against flats in the gear ensure its rotating when the gear rotates.

2.41 When in response to a marking pulse a push bar is lifted by its bell crank, as described in Par. 2.28 of this section, the rocker bail operating blade (see Fig. 15 and 18) engages a slot in the bar and moves it to the left during the first part of the function cycle. The bar, by means of its rack and the mating pinion, rotates the associated eccentric one half revolution where it is locked in position by a detent assembly while printing takes place. When the bail rocks back to the right during the latter part of the cycle, it returns the bar and eccentric to their home position where the eccentric is again detented. The preceding does not apply to the No. 7 pushbar, covered in Par. 2.51. In both assemblies one-half revolution of the rear eccentric results in its maximum vertical displacement which is transferred through the front eccentric to a crank pin. Similarly, one-half revolution of the front eccentric results in its maximum displacement being transferred to the crank pin. If both eccentrics are

rotated, the displacement of the crank pin is equal to the algebraic sum of the two displacements which may be in either the same or opposite directions. Both assemblies are so designed that, if the displacement of the rear eccentric is taken to be one unit, the displacement of the front eccentric is four units. Four permutations are thus available: zero (neither eccentric displaced), one unit (rear eccentric displaced), four units (front eccentric displaced) and five or three units depending on how the assembly is set up (both eccentrics displaced).

2.42 In the right assembly the home position of the rear eccentric is down and the home position of front eccentric is up (Fig. 18). Thus their displacements are in opposite directions — up for the rear and down for the front — and their aggregate displacement is three units downward. Any displacement occurring in the right assembly is imparted to the typewheel rack in equal quantity but opposite direction. For example, if the No. 5 pushbar is selected, it causes the right rear eccentric to be displaced, and one unit of upward motion is transferred through a right output connecting rod to

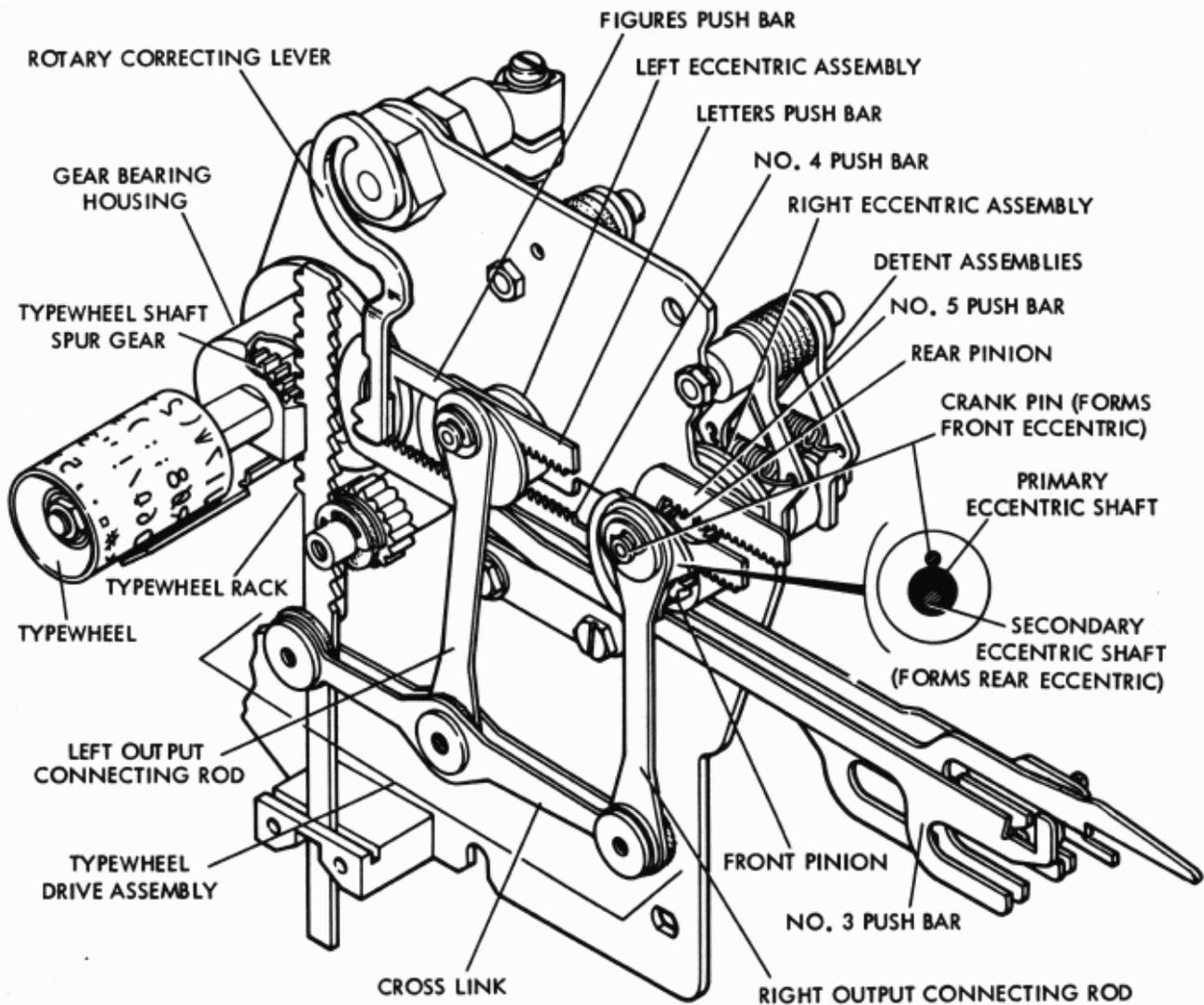


Figure 17. Rotary Positioning Mechanism

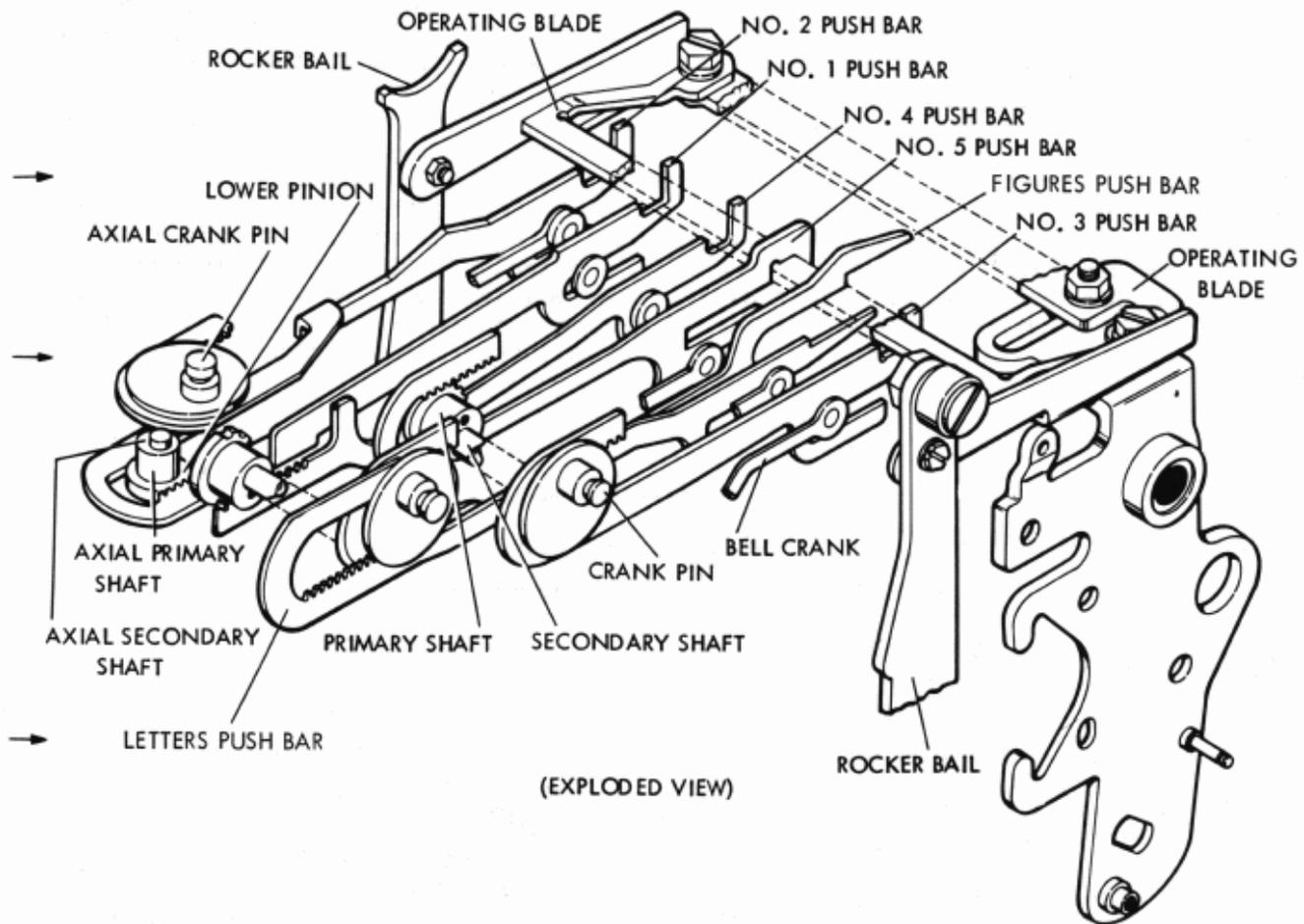


Figure 18. Pushbars and Eccentric Assemblies

the right end of a cross link (Fig. 17). The cross link pivots about a left output connecting rod and at its left end imparts one unit of downward displacement to the typewheel rack. The rack rotates the spur gear, shaft and typewheel one row of characters clockwise from the home position, and the No. 1 clockwise row (Fig. 16) is presented to the print hammer at the time of printing. On its right stroke the No. 5 pushbar returns the eccentric and the typewheel to their home positions. In a similar manner, selection of the No. 3 pushbar results in a four unit downward displacement of the right front eccentric and a four-row, counterclockwise rotation of the typewheel; and selection of both the three and five bars results in a three-row, counterclockwise rotation of the typewheel.

2.43 The home position of the left rear eccentric is up, and any displacement

appearing in the left assembly is transferred to the typewheel rack in double quantity in the same direction. When the No. 4 pushbar is selected, the left rear eccentric is displaced one unit downward. This movement is conveyed through the left output connecting rod to the approximate mid-point of the cross link. The cross link pivots about the right output connecting rod and its left end imparts two units of downward movement to the typewheel rack which rotates the typewheel two rows clockwise from its home position.

2.44 When both eccentric assemblies are displaced, the motion occurring in the typewheel rack is equal to the algebraic sum of the motions resulting from each assembly. For example, if the No. 3, 4 and 5 pushbars are all selected, three units of upward displacement

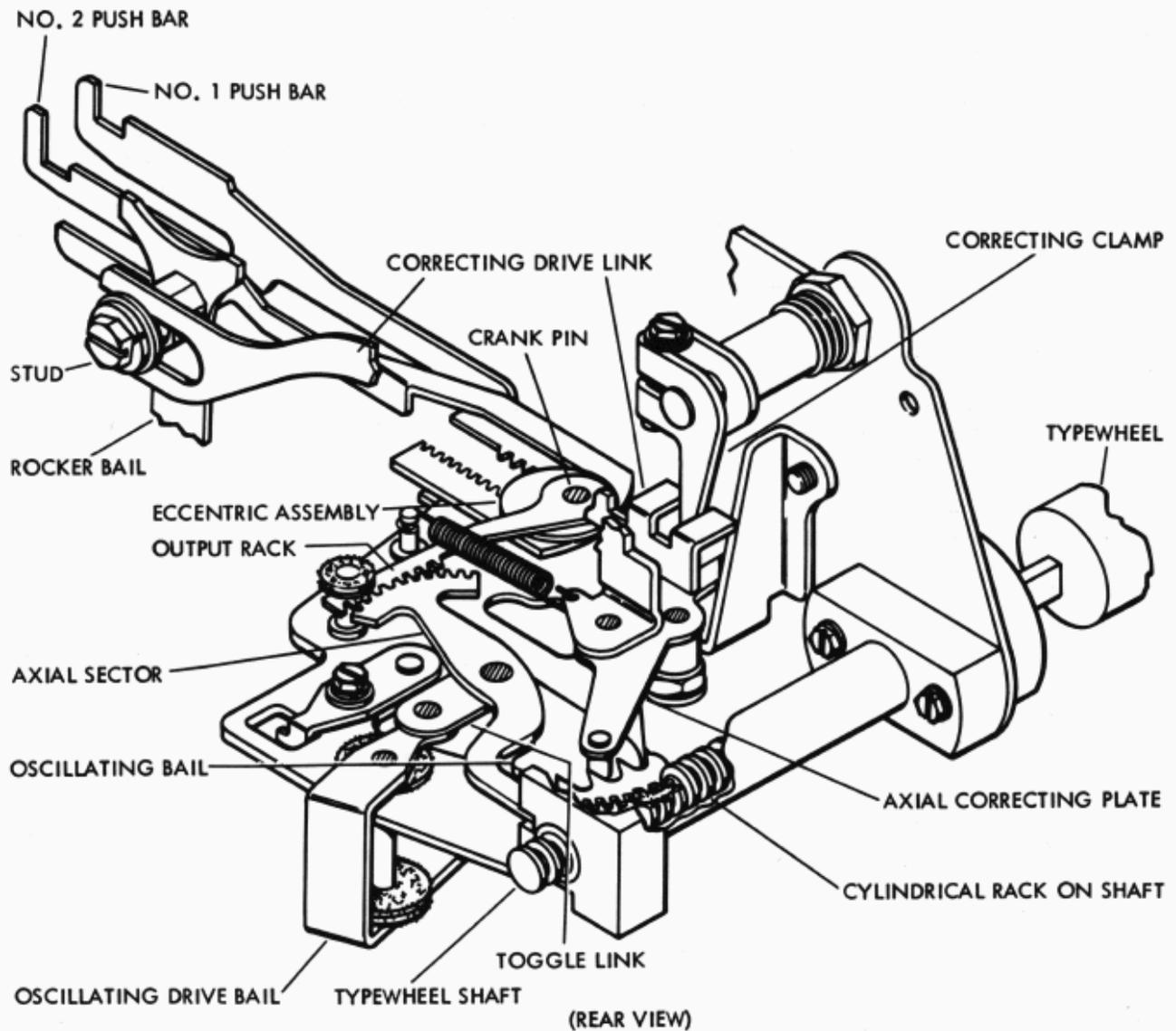


Figure 19. Axial Positioning Mechanism

from the right assembly and two units of downward displacement from the left assembly occur as one unit ($3-2 = 1$) of upward displacement in the rack and a counterclockwise rotation of one row in the typewheel. If neither the No. 3, 4 nor 5 pushbar is selected, the mechanism remains inactive and printing takes place in the No. 0 row. Excluding the left-front eccentric, which is only used for the letters-figures shift, there are eight permutations available in the other three eccentrics, making it possible to select

any of the eight rows in a given section (Fig. 16).

Axial Positioning (Fig. 18, 19 and 21)

2.45 The functions of the axial positioning mechanism are to position the typewheel so that the proper character in the selected row is aligned with the hammer at the time of printing and to retract the typewheel and ribbon guide at the end of the function cycle so that

the last-typed character is visible. The mechanism mounts on an axial bracket supported by the frame and the front plate and includes an eccentric assembly similar to those of the rotary positioning mechanism (Fig. 18 and 19). Two eccentrics, a lower whose pinion is driven by the No. 1 pushbar and upper whose pinion is driven by the No. 2 pushbar, rotate in a horizontal plane in bearing housings attached to the bracket. The eccentric assembly is linked to the typewheel shaft by an axial output rack and sector as shown in Fig. 19.

2.46 The selection of either the No. 1 or No. 2 pushbar results in the maximum displacement toward the rear of the associated eccentric, and the eccentrics are so designed that, if the displacement of the lower is taken to be one unit, that of the upper is two units. Again four permutations are available at the crank pin: zero (neither eccentric displaced), one unit (lower eccentric displaced), two units (upper eccentric displaced) and three units (both eccentrics displaced).

2.47 If during a function cycle neither pushbar is selected, no motion occurs in the axial positioning mechanism with the exception of that resulting from the oscillating assembly (Par. 2.48), and the No. 0 character of the selected row is aligned with the hammer at the time of printing (Fig. 16). On the other hand, if the No. 1 pushbar is selected, it causes the lower eccentric to revolve and one unit of displacement to be transferred by the crank pin to the axial output rack. The rack moves to the rear and passes the motion to the axial sector which pivots counterclockwise (as viewed from above). The right end of the sector, by means of a cylindrical rack in the typewheel shaft, moves the typewheel one character forward from its home position. The No. 1 character is printed, and when the pushbar reverts to its unselected position it returns the axial linkage and typewheel to their home position. If the No. 2 pushbar is selected the No. 2 character is printed, and if both pushbars are selected, the No. 3 character is printed. The cylindrical rack has no lead, and the shaft can thus be rotated while being moved axially.

2.48 With each cycle of the function clutch, an oscillating drive link transfers from the rocker bail an unselected motion to an oscillating drive bail (Fig. 19 and 21). This movement is passed by toggle links to an oscillating bail and the sector pivot. The effect of

this action is to introduce a separate motion to the sector tending to cause it to pivot about the teeth on the output rack. During the fore part of the function cycle, if no axial pushbar is selected, the right end of the sector is moved forward slightly and positions the No. 0 character for printing. At the end of any cycle the sector retracts the typewheel slightly so that the last printed character is visible. Concurrent with the above operation, a ribbon oscillating lever is made to pivot about its left end and with each cycle project and retract the ribbon guide which would obstruct the view of the character (Fig. 21).

Correction (Fig. 17 and 19)

2.49 After the typewheel has been positioned by the axial and rotary positioning mechanisms, the selected character is more accurately aligned for printing by the correcting mechanism which compensates for any play and backlash in the positioning linkages. Each function cycle the rocker bail transfers motion through a correcting drive link to a correcting clamp and shaft (Fig. 19). The shaft pivots a rotary correcting lever (Fig. 17) which is equipped with an indentation that engages a tooth in a typewheel rack. There is a tooth in the rack for each row of characters (16 in all), and they are so correlated with the typewheel that when a tooth is engaged by the corrector its row is accurately aligned with the print hammer. Axial correction, which is accomplished simultaneously, is similar to rotary correction: the drive link rotates an axial correcting plate counterclockwise (as viewed from the above), and a roller mounted on the plate engages a notch in the axial sector (Fig. 19). Thus the typewheel is accurately aligned in both fields of motion just before printing takes place. During the latter part of the function cycle, a correcting drive link spring returns the correcting mechanism to its home position.

2.50 Since the rocker bail is the source of motion for both the pushbars and the positioning mechanisms, correction must take place at a point near enough to the extreme travel of the bail that it does not interfere with the movement of the typewheel rack or axial sector. In addition, because the rocker bail controls the tripping of the print hammer, which occurs very late in the bail's stroke, it becomes necessary to utilize the time between the tripping of the hammer and its striking the paper to accomplish correction. The delay

in actuating the correcting mechanism is effected by allowing a drive stud on the rocker bail to slide in an elongated slot in the correcting drive link during the early part of the cycle.

Typewheel Shift (Fig. 17 and 20)

2.51 The typewheel shift from the letters to the figures printing segment (or figures to letters) is controlled by the No. 7 selector push lever through an associated train of levers in the transfer mechanism and two pushbars which engage a common pinion. The pushbars are connected to a common bell crank which is, in turn, controlled by the No. 7 pulse beam and transfer lever.

2.52 To shift the typewheel from the figures section to the letters section, a marking No. 7 pulse must be received by the unit. This will cause the No. 7 punch slide to be selected and move to the left (Par. 2.23). As the No. 7 punch slide moves left, it rotates its associated transfer lever counterclockwise which, in turn, pivots the No. 7 pulse beam clockwise. This allows the associated bell crank to rotate counterclockwise, under spring tension, and lift the letters-figures pushbars until the step on the end of the letters pushbar is raised to a height which will bring it into engagement with the rocker bail operating blade, when the blade moves to the left (Par. 2.34). The operating blade simultaneously pushes the letters pushbar to the left and the figures pushbar to the right, resulting in rotation of the typewheel to the letters section. As long as the No. 7 pulse is marking, the letters pushbar will remain in this left-most position.

2.53 When the No. 7 pulse changes from marking to spacing, the punch slide will remain unselected, and the pushbars will not be lifted by the bell crank-transfer lever linkage. The figures pushbar, which is furthest to the right, will then be in such a position that the step on its end extension will be engaged (and pushed) by the rocker bail operating blade as the blade moves to the left, resulting in rotation of the typewheel to the figures position. As the figures pushbar moves left, the letters pushbar simultaneously moves to the right.

2.54 As long as the No. 7 pulse is spacing, the letters-figures pushbars will not be lifted and, therefore, the letters pushbar will not be moved to the left (Par. 2.52). The typewheel will shift back to the letters section only upon receipt of a No. 7 marking pulse by the reperforator.

C. Printing (Fig. 21)

2.55 After the typewheel has been positioned and corrected, the printing mechanism supplies the impact which drives the paper and ribbon against the selected character. It effects this operation by means of a print hammer which is mounted on a shaft supported by a bracket attached to the typewheel bearing housing. In its unoperated condition, as illustrated in Fig. 21, the hammer is held against an accelerator by a relatively weak spring. The accelerator is mounted on the hammer shaft and is retained by a printing latch in its upper position against the tension of a relatively strong spring.

2.56 The rocker bail, during the fore part of the function cycle, moves a printing drive link to the right (as viewed from the rear in Fig. 21) and causes a pivot arm to rotate clockwise. The arm lowers a trip link which slides in an elongated slot. Near the end of the rocker bail's travel, the trip link pivots the latch which releases the accelerator. Under the spring tension, the accelerator snaps down and impels the hammer upward. The face of the hammer drives the tape and inked ribbon up against the typewheel and imprints the selected character on the tape. The imprint may be in black or red, as determined by the ribbon shift mechanism (Par. 2.60). Near the end of its travel, the accelerator encounters a projection on a latch bracket, and inertia carries the hammer the rest of the way. As the rocker bail returns to its home position, it causes the trip link to move up, release the latch and return the accelerator to its latched position.

D. Ribbon Feeding (Fig. 22)

2.57 The characters are typed in red or black ink supplied by an inked two-color ribbon which is held between the tape and the typewheel by a guide and advanced by the ribbon feed mechanism (Fig. 22). The path of the ribbon is down to the right off the top of a right spool, under a right roller, through right pins on the reversing arm, through the guide, up through left pins on the reversing arm, over a left roller, and to the right over the top of a left spool.

2.58 Each function cycle, as the rocker bail nears the end of its left travel, a roller mounted on its forward arm pivots a drive arm clockwise. The drive arm lifts a feed pawl which advances the ribbon by rotating a ratchet on one of the ribbon spools one tooth. A retaining pawl under spring tension detents the

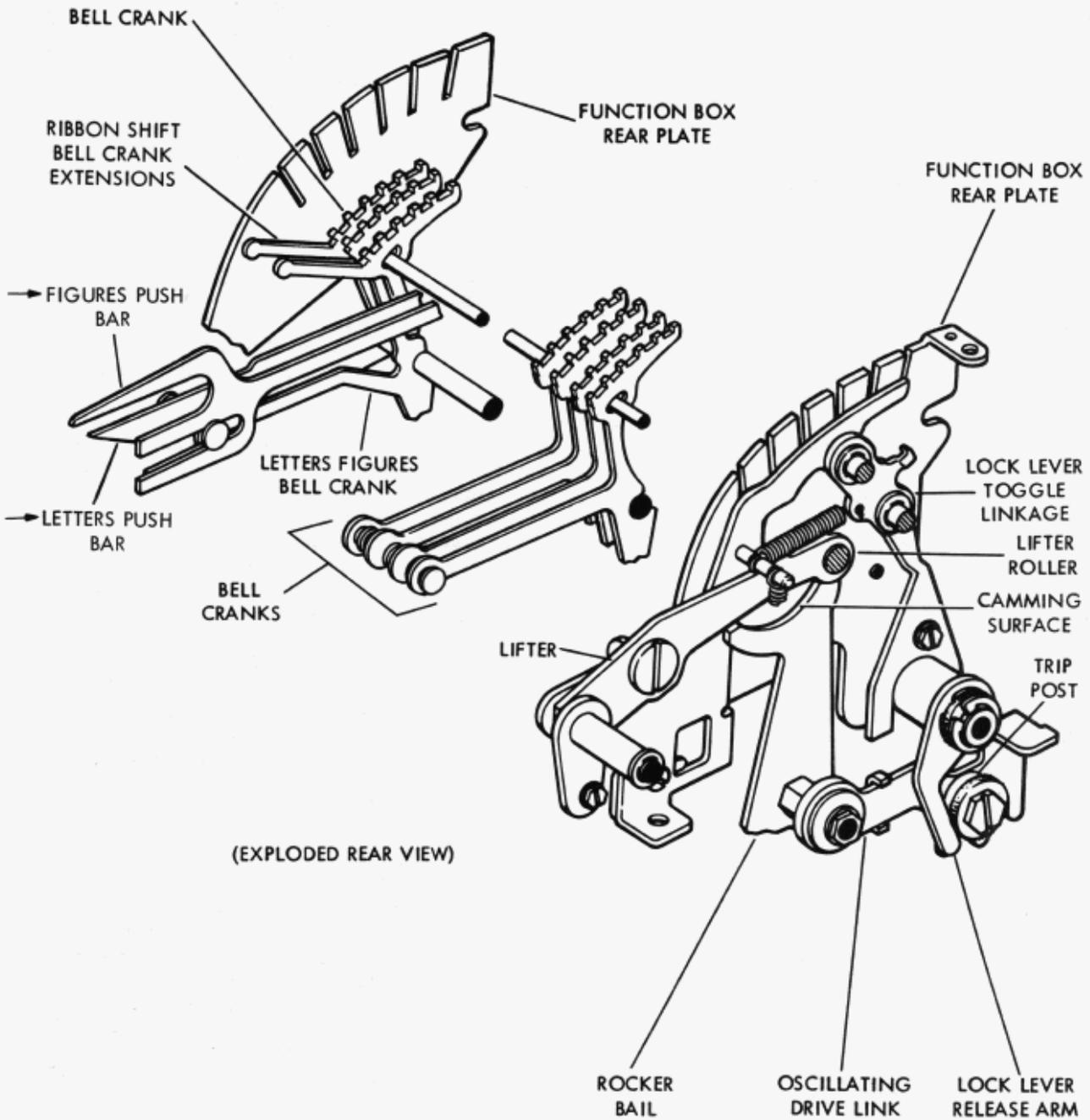


Figure 20 - Function Box

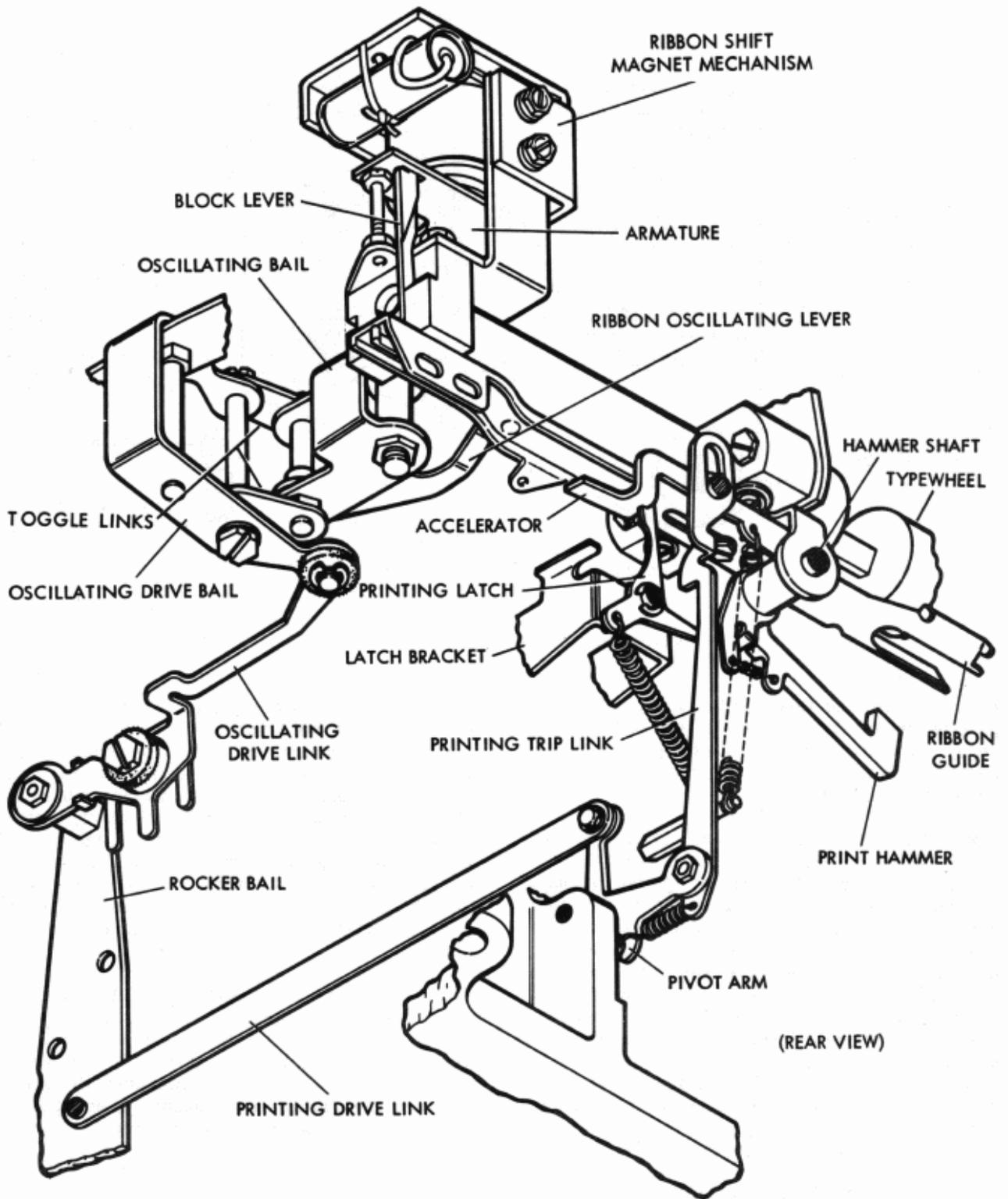


Figure 21 - Printing Mechanism

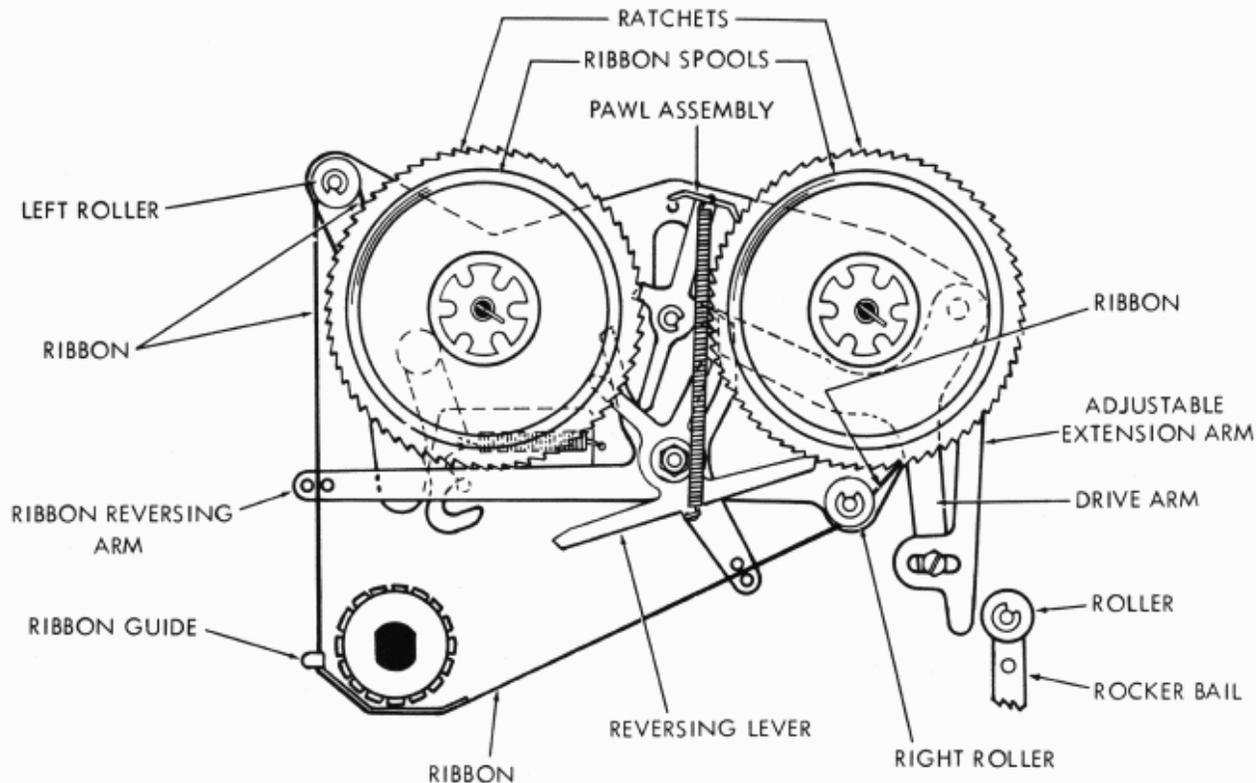


Figure 22 - Ribbon Feed Mechanism

ratchet while the feed pawl, during the latter part of the function cycle, is lowered so as to engage the next tooth. Each operation, the ribbon is advanced in this manner until the ribbon feed mechanism is reversed.

2.59 When a spool is almost depleted, a rivet in the ribbon encounters pins on the reversing arm, and the stress applied through the ribbon as it is rolled on the other spool pivots the arm. As the pawl assembly is lowered at the end of the next operation, an extension strikes the reversing arm, and the pawl is shifted against the other ribbon spool ratchet. The pawl's rounded lower extension pivots a reversing lever which shifts the retaining pawl so that it engages the opposite ratchet. The ribbon will then feed in the opposite direction until again reversed. A detent holds the reversing arm in position until its next reversal.

E. Ribbon Shift Mechanism (Fig. 21)

2.60 As the ribbon carrier drive arm is driven by the motion of the axial oscillator lever, the ribbon carrier follows by the action of a

spring. If the ribbon color shift magnet is energized and its armature attracted, a block lever is raised into the path of the ribbon carrier and blocks the carrier from any further motion. In this case, a black ribbon is over the typewheel, causing a black character to be printed. If, on the other hand, the ribbon color shift magnet is not energized, the ribbon carrier is allowed to follow its operating arm, and the red portion of the ribbon is positioned over the print hammer, resulting in a red character.

2.61 When the No. 6 and 7 signal bits are the same, both marking or both spacing, the ribbon shift magnet is de-energized, and a red character is printed. If, however, the No. 6 and 7 bits are different, one marking, the other spacing, the ribbon shift magnet is energized, and a black character will be printed.

TAPE PERFORATING AND FEEDING (Fig. 23)

A. General

2.62 The perforating mechanism rolls the tape between a feed wheel and a die wheel,

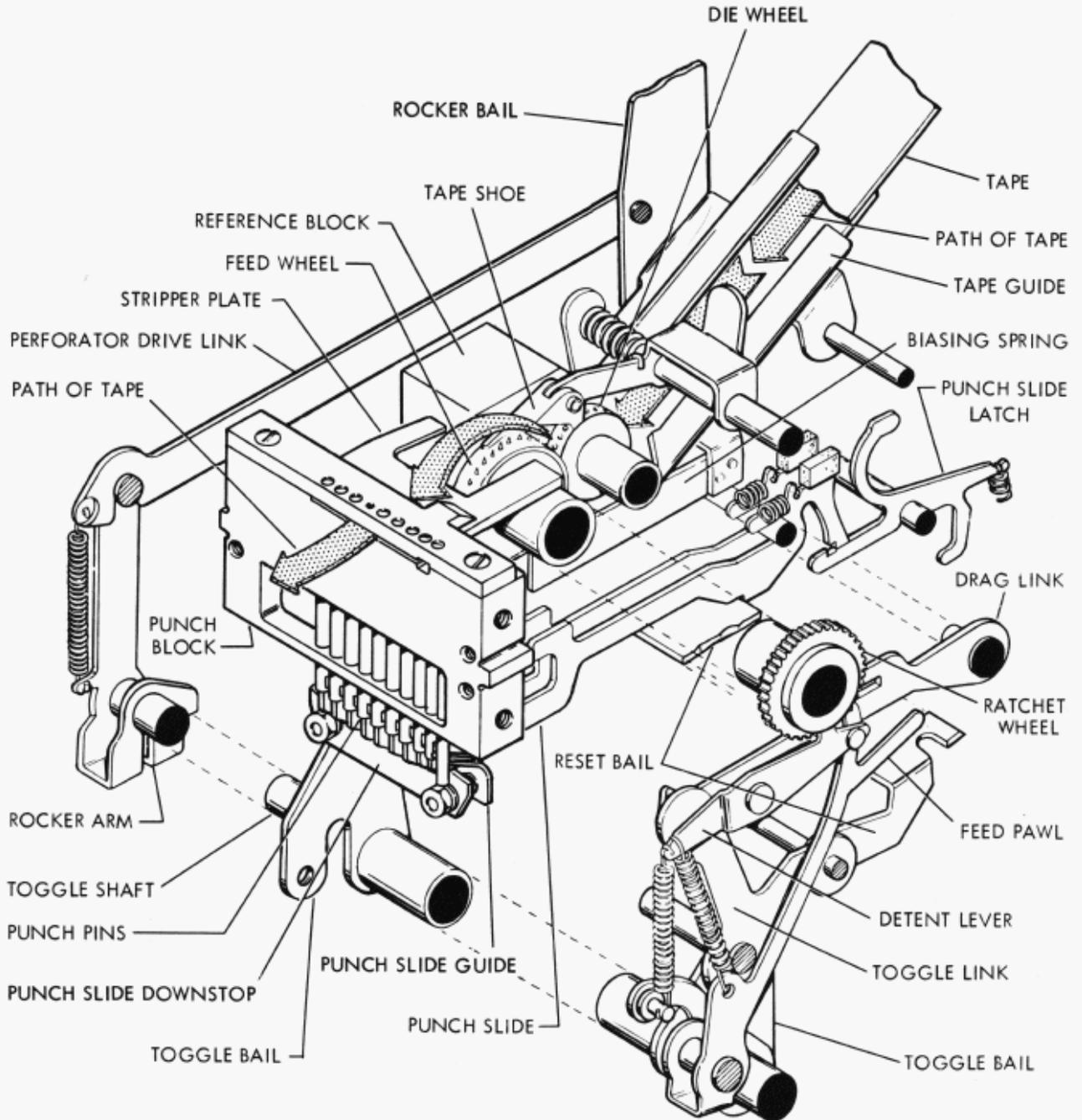


Figure 23 - Perforating Mechanism

which does not perforate the feed hole but merely regulates the amount of tape feed. The punch perforates round holes corresponding to the code combination received from the signal line and perforates a smaller feed hole positioned between the third and fourth intelligence levels. Intelligence is received from the selecting mechanism by the punch slides, which select the proper punch pins in a punch block assembly (Fig. 23). Motion from the rocker bail is distributed to the pins and the tape feed parts by a main bail assembly, which includes a toggle bail, a toggle shaft, a slide post, toggle links, drag links and the punch slide reset bail.

B. Perforating

2.63 As described in Par. 2.32 near the end of the selecting cycle, the reset bail is lowered and releases the eight punch slides (Fig. 13). The selected slides move to the left, and the unselected slides are retained to the right by their latches. In the selected position, a projection of each slide extends over the slide post. Since a feed hole is perforated every operation, the punch slide associated with the feed hole punch pin is designed so that it is always in a selected position. During the first part of the function cycle, the rocker bail moves to the left and, by means of a drive link and rocker arm, rotates the toggle shaft and bail counterclockwise. Toggle links attached to the front and rear of the bail lift the slide post and move the reset bail to the left. The selected slides are carried upward by the post and force the associated pins through the tape. The slides thus become an integral part of the main bail assembly during the perforating stroke. Approximately midway through the function cycle, the function trip assembly lifts the reset bail.

2.64 During the last half of the cycle, the toggle bail is rotated clockwise, pulling the slide post down and lowering the selected punch slides. The punch slides, which engage notches in their respective punch pins, pull the punch pins down below the tape. The main bail assembly and the selected punch slides and their associated punch pins move as a unit during the perforating stroke, both up and down. The punch pins are positively driven and retracted, to produce the fully perforated tape.

2.65 A chad chute, mounted on the reperfocator punch block, mates with a chute on the mounting base. The chutes carry chad punched from the tape into a chad container on the

tape handling stand. Refer to the appropriate section for a detailed discussion of the chad storing mechanisms.

C. Feeding

2.66 Tape feeding is accomplished after perforation during the last half of each function cycle. The tape is threaded down through a tape guide and then up between a feed wheel and die wheel (Fig. 23). A feed pawl, driven by the toggle bail, acts upon a ratchet and rotates the feed wheel which, by means of sharp pins and a slot in the die wheel, advances the tape one character at a time. A detent with a roller that rides on the ratchet holds the feed wheel and tape in position during perforation. The detent and feed pawl springs are so positioned that the pressure of the detent on the ratchet is high during the first half of the perforation, but is low during idling and the last half of the cycle to facilitate tape threading and feeding. A tape shoe retains the tape on the feed wheel, and a biasing spring holds it back against a reference block, so that the feed holes are punched a constant distance from the edge. The tape is stripped from the feed wheel by a stripper plate, passes into the punch block, where it is perforated, and finally emerges at the left.

→ 3. VARIABLE FEATURES

NON-INTERFERING TAPE FEED-OUT (Fig. 12, 14 and 24)

A. Initiation Mechanism (Fig. 24)

→ 3.01 The feed-out operation is initiated by an external pulse (115 V DC \pm 10 percent with 600 ohms series resistance). The pulse is applied to the feed-out magnet when the reperfocator is in an idling condition. When the magnet is energized, the armature bail moves the blocking bail out of engagement with the drive bail assembly. The drive bail, which is spring loaded, falls into the indent of its cam, and the connecting link positions the release lever on the lower step of the latch lever. If the start magnet is held energized longer than one cycle, the non-repeat latch prevents the drive bail from again falling into the indent of its cam. The non-repeat latch is delayed one cycle by the spring loaded blocking latch on the drive bail. As the drive bail reaches the indent of its cam, the blocking latch rides over the non-repeat latch. The drive bail then reaches the high part of its cam, and the non-repeat latch falls into engagement with the drive bail. When the start magnet is de-energized, the spring loaded blocking bail again engages the drive bail and simultaneously disengages the non-repeat latch.

B. Metering Mechanism (Fig. 24)

3.02 When the drive bail positions the release levers on the lower step of the latch lever as described above (Par. 2.67), metering takes place. The release lever has now permitted the check pawl and feed pawl to engage two adjacent ratchets. One of the ratchets is fed continually by the feed pawl. This ratchet has a deeper notch at every sixth tooth, so that the pawl engages the second ratchet on every sixth cycle. After the second ratchet has rotated an amount equivalent to two teeth, a follower, riding a cam attached to the ratchet, drops off its peak and unblocks the tripping mechanism. After a predetermined length of tape has been fed (as measured by the second ratchet), the latch lever is actuated, as it

would be by the selector cam on receipt of a message, and the tripping mechanism is blocked to prevent further feeding. Simultaneously, the feed pawls are lifted off the ratchets, and the ratchets return to their zero position.

C. Tripping Mechanism (Fig. 12)

3.03 A bail that follows a cam attached to the reperforator main shaft engages the clutch trip lever and punch slide latches. When the bail cam follower enters the detent of its cam, the bail operates the function clutch trip lever and punch slide latches, initiating a RUBOUT cycle of the printing and perforating mechanisms. Each time the reperforator main shaft rotates one revolution, a RUBOUT cycle is initiated, providing the bail is not blocked by the

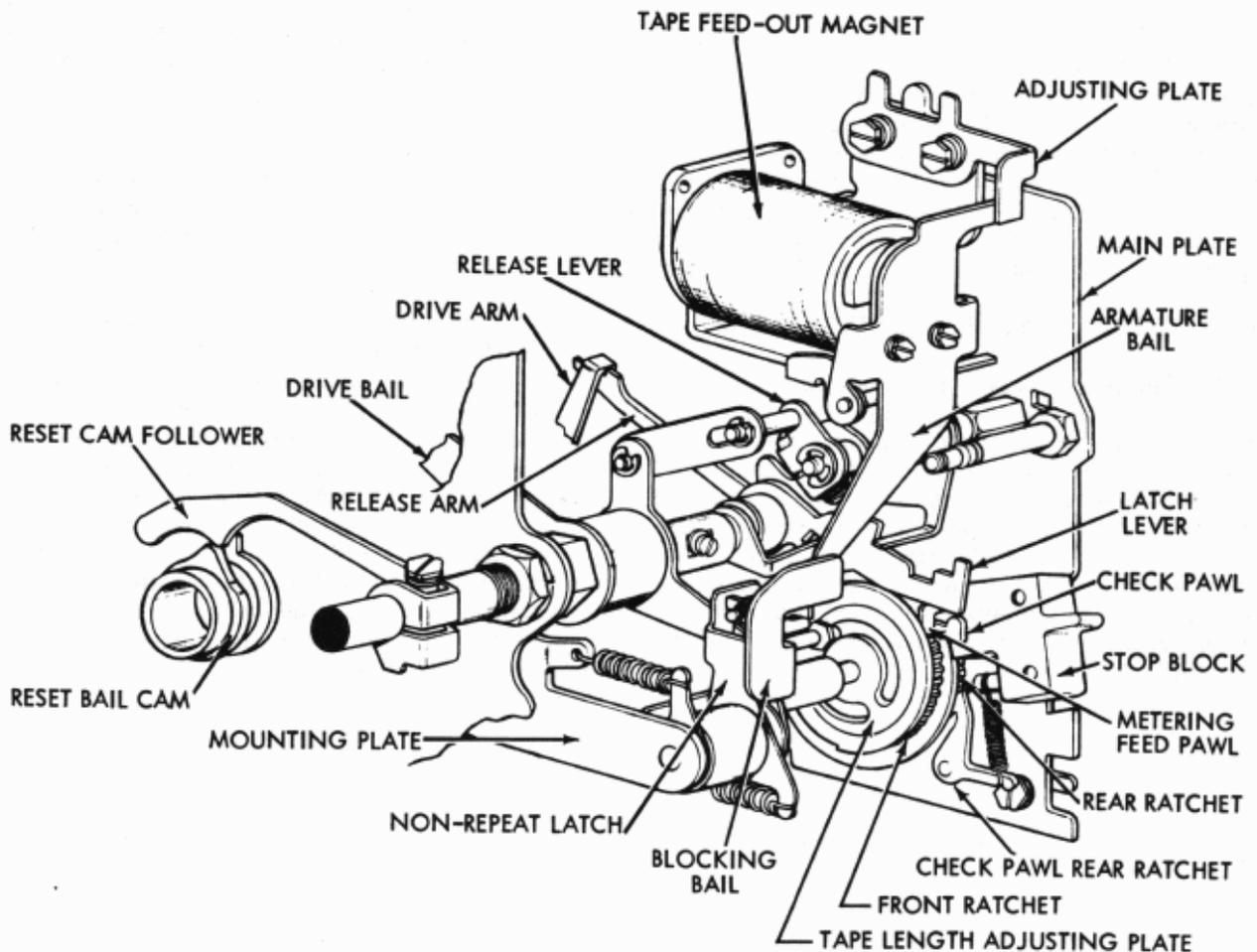


Figure 24 - Non-Interfering RUBOUT Tape Feed-Out Mechanism (Remote Control)

latch lever (Par. 2.70). If an incoming message trips the latch lever, the RUBOUT function cycle is immediately blocked from any further operation.

D. Storage Mechanism (Fig. 14)

- 3.04 The purpose of the storage mechanism is to hold the reset bail (perforating mechanism) in engagement with the slides until the slides are fully reset, so that they may recognize the first character set up in the punch slide latches by the selecting mechanism. This mechanism consists of a latch that is operated by a link attached to the punch slide reset bail toggle. During reception of an incoming message, the toggle mechanism pushes the latch out of the way of the reset bail prior to its being stripped by the clutch trip lever.

→ E. Timing Contacts

3.05 A pair of contacts mounted on the left side of the feed-out mechanism main plate at the rear of the reperforator may be wired to either make or break an external circuit when the feed-out mechanism is operating. An extension of the feed-out mechanism latch lever rotates the timing contact bail to open the normally closed contacts and close the normally open contacts in step with the feed-out cycle.

POWER DRIVE BACKSPACE MECHANISM

- 3.06 The power drive backspace mechanism is used to reverse the tape feed to delete errors in perforated information. The application of an external pulse (a.c. or d.c.) initiated by the backspace key on an associated keyboard unit moves the perforated tape one code space to the right with each pulse. The mechanism consists of a magnet located beneath the punch mechanism, an eccentric drive on the front end of the main shaft, and a feed assembly at the → punch block.

- 3.07 When the magnet is energized, the armature bail is pulled downward. An extension on the bail disengages a drive link latch, which drops, engaging a notch on the eccentric arm. As the main shaft moves the eccentric arm to the left, a bell crank is depressed, contacting the perforator feed pawl and disengaging it. The backspace feed pawl engages the feed wheel ratchet and rotates the feed mechanism counterclockwise. When the magnet is de-energized, the drive link is disengaged from the eccentric arm, which slides freely along the pivot post of the drive link.

PRINT SUPPRESSION MECHANISMS

3.08 Manual and automatic print suppression operate similarly to block the movement of the print hammer and prevent contact between the message tape, inked ribbon and type-wheel. Manually controlled suppression operates through a lever extending from the front of the reperforator at the base of the punch pins.

3.09 Manual print suppression is accomplished by raising the NO PRINT lever at the front of the reperforator. This rotates a blocking extension across the top of the print hammer, preventing all printing, regardless of the input code.

- 3.10 Automatic printing suppression is a function controlled by the ribbon shift mechanism. Primarily used with typing reperforators with one color ribbons, automatic printing suppression is operative on the same code combinations which, in the basic unit, initiate the red ribbon printing shift. An accelerator blocking link attached to the ribbon carrier prevents the print hammer accelerator from rotating downward when the release latch is disengaged. As a result, printing is suppressed whenever a no-current condition keeps the ribbon shift blocking link engaged with the ribbon carrier.